

B.1 Gear Calculations

11-07-2006 Spur Gears Component Generator (Version 10.0.0)

--- Guide

External Gearing - ANSI

Calculation of geometry: Calculates the center distance according to the diametral pitch, number of teeth, correction and helix direction

Distribution of Correction: User

Load calculation: Calculates the torque according to the power and speed

Method of Strength Calculation: Strength check calculation

--- Basic Parameters

Desired Gear Ratio = 16.5

Pressure Angle $\alpha = 20^\circ$

Addendum $a^* = 1$ (= 0.0139 in)

Clearance $c^* = 0.25$ (= 0.0035 in)

Root Fillet = 0.38 (= 0.0053 in)

Addendum of Basic Rack = 1.25 (= 0.0174 in)

Helix Angle $\beta = 0^\circ$

Diametral Pitch $P = 72$ /in

Center Distance $a_w = 2.1875$ in

Product Center Distance $a = 2.1875$ in

Total Unit Correction = 0

Operating Pressure Angle $\alpha_{haw} = 20^\circ$

Circular Pitch $p = 0.0436$ in

Base Circular Pitch $p_{tb} = 0.041$ in

Contact Ratio = 1.6447 (1.6447 + 0)

Precision Specification 10

Limit Deviation of Helix Angle $F_\beta = 0.00135$ in

Limit Deviation of Axis Parallelity $f_x = 0.00135$ in

Limit Deviation of Axis Parallelity $f_y = 0.0007$ in

--- Gear 1

Number of Teeth = 18

Unit Correction = 0.26 (= 0.0036 in)

Pitch Diameter $d = 0.25$ in

Base Circle Diameter $d_b = 0.2349$ in

Outside Diameter $d_o = 0.285$ in

Root Diameter $d_f = 0.2225$ in

Work Pitch Diameter $d_w = 0.25$ in

Tooth Thickness $s = 0.0244$ in

Outside Tooth Thickness = 0.568 (= 0.0079 in)

Facewidth = 0.1875 in
Face width Ratio = 0.5
Chordal Thickness T = 0.0216 in

Chordal Thickness Height ht = 0.0136 in
Dimension Over (Between) Wires M = 0.3068 in

Wire Diameter dw = 0.03 in
Limit Circumferential Run-out Fr = 0.002 in
Limit Deviation of Axial Pitch fpt = ± 0.00105 in
Limit Deviation of Basic Pitch fpb = ± 0.001 in

--- Gear 2

Number of Teeth = 297
Unit Correction = -0.26 (= -0.0036 in)
Pitch Diameter d = 4.125 in
Base Circle Diameter db = 3.8762 in
Outside Diameter do = 4.1456 in
Root Diameter df = 4.0831 in
Work Pitch Diameter dw = 4.125 in
Tooth Thickness s = 0.0192 in
Outside Tooth Thickness = 0.837 (= 0.0116 in)
Facewidth = 0.125 in
Face width Ratio = 0.0303
Chordal Thickness T = 0.0169 in

Chordal Thickness Height ht = 0.0072 in
Dimension Over (Between) Wires M = 4.1751 in

Wire Diameter dw = 0.03 in
Limit Circumferential Run-out Fr = 0.002 in
Limit Deviation of Axial Pitch fpt = ± 0.00105 in
Limit Deviation of Basic Pitch fpb = ± 0.001 in

--- Load (Gear 1; Gear 2)

Power P = 0.0003; 0.0003 HP
Efficiency = 0.97
Speed n = 120; 7.2727 rpm
Torque Mk = 0.0131; 0.2102 lb ft
Tangential Force Ft = 1.2605 lb
Radial Force Fr = 0.4588 lb
Axial Force Fa = 0 lb
Normal Force Fn = 1.3414 lb
Circumferential Speed v = 0.1309 ft/s
Resonance Speed nE1 = 203904.13 rpm

Strength Check According to ANSI

Durability $L_h = 8$ hour

--- Material Values

&Material designation: T300 sst; 6061-T6

Endurance Limit $S_n = 34800$; 18000 psi

Surface Fatigue Strength $S_{fe} = 43500$; 28000 psi

Modulus of Elasticity in Tension [10^3] = 28000; 10500 psi

Poisson's Ratio = 0.25; 0.33

--- Factors for Bending

Overload Factor $K_o = 1$

Dynamic Factor $K_v = 1.2$

Mounting Factor $K_m = 1.4$; 1.4

Load Factor $C_l = 1$

Gradient Factor $C_g = 1$

Surface Factor $C_s = 0.8$; 0.8

Reliability Factor $k_r = 0.814$

Temperature Factor $k_t = 1$

Mean Stress Factor $k_{ms} = 1.4$; 1.4

Geometry Factor $J = 0.28$; 0.5

--- Factors for Contact

Overload Factor $K_o = 1$

Dynamic Factor $K_v = 1.2$

Mounting Factor $K_m = 1.4$; 1.4

Life Factor $C_{li} = 1.451$; 1.887

Reliability Factor $C_r = 1$; 0.8

Elastic Factor $C_p = 1640$

Geometry Factor $I = 0.1515$

--- Calculation Results

Factor of Safety from Tooth Breakage $k_n = 8.901$; 6.727

Factor of Safety from Pitting $k_f = 1.82$; 1.219

Strength Check - True