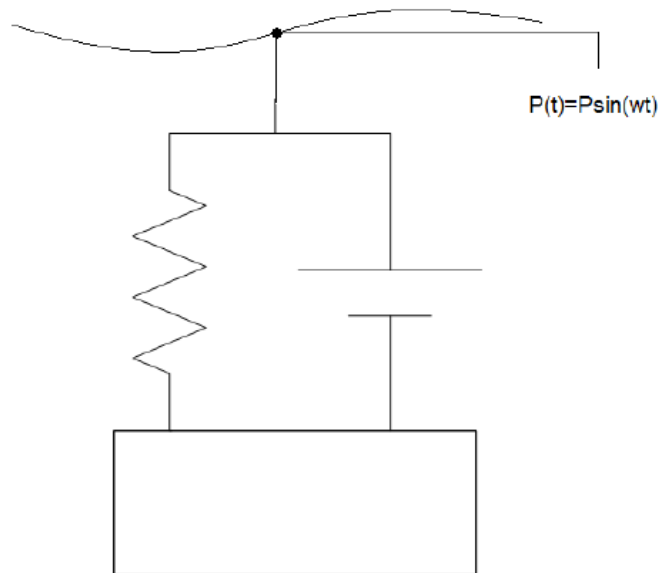


B.2 Vibration Calculations

Vibration Analysis

The motion is in the vertical direction only
The motion at point P is the input to the system
The vertical motion $x(t)$ of the body is the output



Equations of Motion

$$m\ddot{x} + b(\dot{x} - \dot{p}) + k(x - p) = 0$$

$$m\ddot{x} + b\dot{x} - b\dot{p} + kx - kp = 0$$

$$m\ddot{x} + b\dot{x} + kx = b\dot{p} + kp$$

$$(ms^2 + bs + k)X(s) = (bs + k)P(s)$$

$$\frac{X(s)}{P(s)} = \frac{b \cdot s + k}{ms^2 + b \cdot s + k}$$

Sinusoidal Transfer Function

$$\frac{X(j\omega)}{P(j\omega)} = \frac{bj\omega + k}{-m\omega^2 + bj\omega + k}$$

$$TR = \frac{\text{amplitude_of_the_system}}{\text{amplitude_of_the_output}}$$

$$TR = \frac{|X(j\omega)|}{|P(j\omega)|} = \frac{\sqrt{b^2 \cdot \omega^2 + k^2}}{\sqrt{(k - m\omega^2)^2 + b^2 \omega^2}}$$

$$\frac{k}{m} = \omega_n^2$$

$$\frac{b}{m} = 2 \cdot \zeta \cdot \omega_n$$

Transmissibility is in terms of the damping ratio and the undamped natural frequency ω_n and $\beta = \omega / \omega_n$

$$TR = \frac{\sqrt{1 + (2 \cdot \zeta \cdot \beta)^2}}{\sqrt{(1 - \beta^2)^2 + (2 \cdot \zeta \cdot \beta)^2}}$$

Calculating the values of each component:
 We will assume that the system will be a total of 40lb

Converting from English to SI units

$$40\text{lb} = 18.144\text{kg}$$

Spring Constant

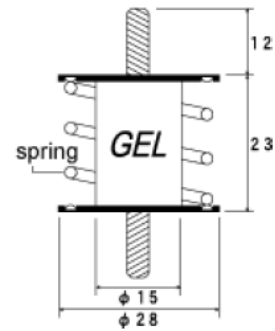
$$k := 16600 \frac{\text{N}}{\text{m}}$$

$$k = 1.66 \times 10^4 \frac{\text{N}}{\text{m}}$$

$$b := 4000 \frac{\text{N}\cdot\text{s}}{\text{m}}$$

$$\omega := 69.115 \frac{\text{rad}}{\text{s}}$$

$$m := 18.144\text{kg}$$



harmonic displacement

$$x := .002\text{m}$$

$$45 \frac{\text{lb}}{\text{in}} = 803.609 \frac{\text{kg}}{\text{m}}$$

natural frequency

$$\omega_n := \sqrt{\frac{k}{m}}$$

damping ratio

$$\zeta := \frac{b}{2 \cdot m \cdot \omega_n}$$

$$\zeta_1 := 0.5$$

frequency ratio

$$\beta := \frac{\omega}{\omega_n}$$

$$\beta = 2.285$$

$$TR := \frac{1 + (2 \cdot \zeta \cdot \beta)^2}{\sqrt{(1 - \beta^2)^2 + (2 \cdot \zeta \cdot \beta)^2}}$$

Solving for the Amplitude of absolute acceleration

$$X := x \frac{1 + (2 \cdot \zeta \cdot \beta)^2}{\sqrt{(1 - \beta^2)^2 + (2 \cdot \zeta \cdot \beta)^2}}$$

$$X = 2.428 \times 10^{-3}$$

Solving for the displacement of the projector relative to the support

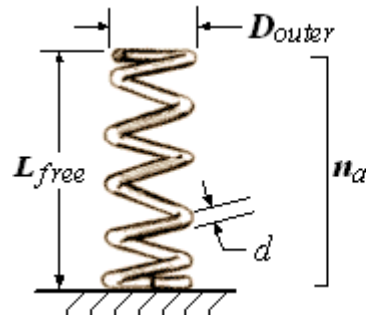
$$Z := x \frac{\beta^2}{\sqrt{(1 - \beta^2)^2 + (2 \cdot \beta \cdot \zeta)^2}}$$

Displacement $Z = 7.597 \times 10^{-4}$

The relative displacement to the support in the Z direction is 0.76 mm which is a large improvement.

This calculation was done at www.efunda.com

In determining the [total number of coils](#) in the spring, the calculator assumes that the ends of the spring are squared.



Material Data: E, ν, ρ
 $k, F_{max}, \tau_{max}, \dots = ?$

Inputs

Diameter of spring wire, d :	3	mm
Outer diameter of spring, D_{outer} :	28	mm
Free length of spring, L_{free} :	23	mm
Number of active coils, n_a :	3	
Youngs modulus of material, E :	200	GPa
Poisson ratio of material, ν :	0.3	
Density of material, ρ :	7500	kg/m ³

Answers

Spring constant, k :	1.66×10^4 N/m	N/m
Maximum load possible, F_{max} :	133 N	N
Maximum shear stress possible, τ_{max} :	3.69×10^5 kPa	kPa
Maximum displacement possible, L_{def} :	0.800 cm	cm
Length of wire required to make spring:	39.4 cm	
Solid height:	1.50 cm	
Distance between coils in free spring:	0.767 cm	
Rise angle of coils:	5.58 deg	
Lowest spring resonant frequency, f_{res} :	446 Hz	Hz
Shear modulus of material, G :	76.9 GPa	GPa
Mass of spring:	0.0209 kg	kg

Select desired output units for next calculation.

Program: CCTT

TestIdent: Weapon MK-19 Continuous

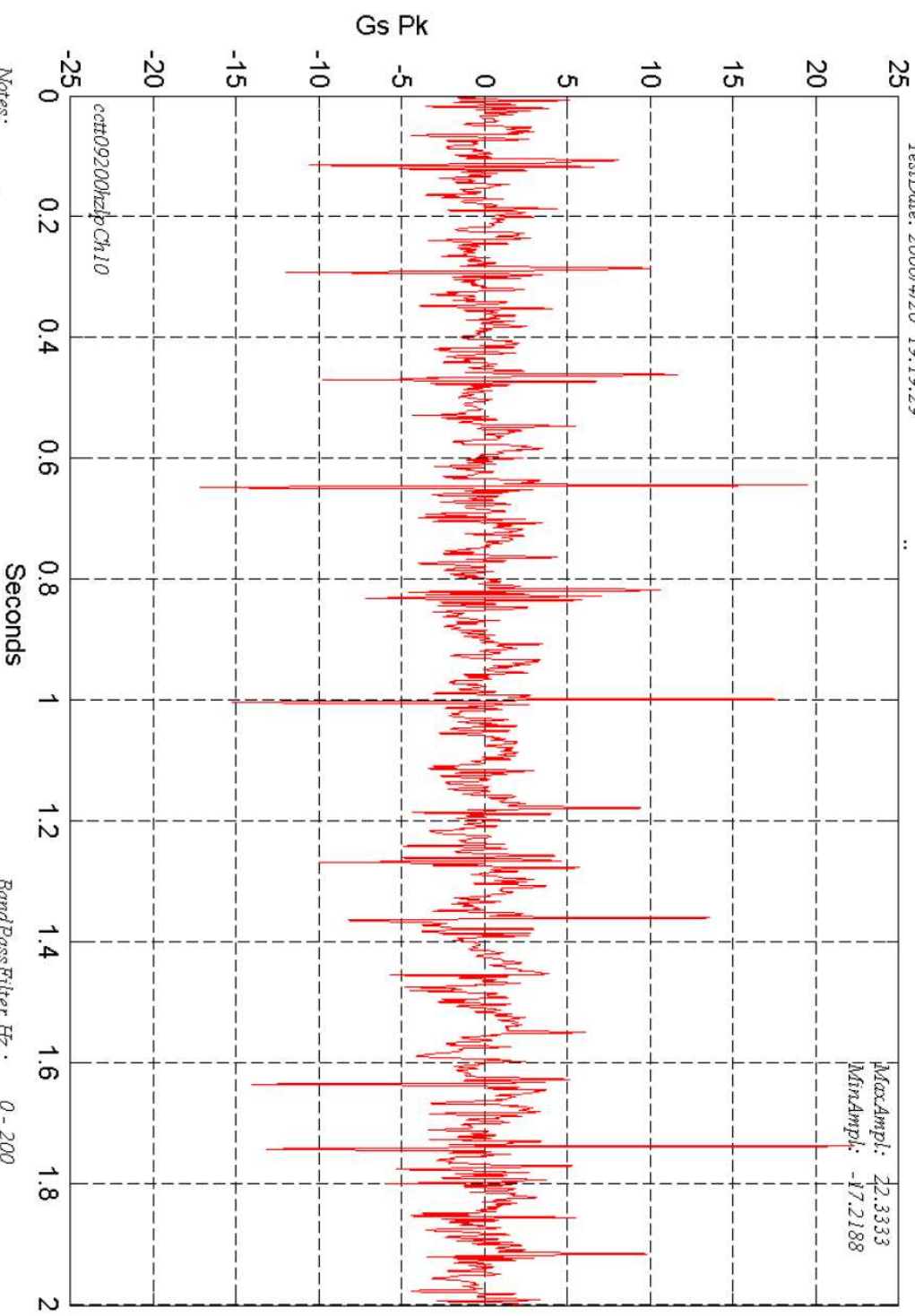
Time History Analysis

TestDate: 2006/4/20 19:19:29

TestCond : 200 Hz Bandwidth

Resp.Id: WeapPntBaseZ

MaxAmpl: 22.3333
MinAmpl: -17.2188



Notes:
Notes:

BandPassFilter Hz : 0 - 200
SampleRate S/Sec : 8192

Program : CCTT

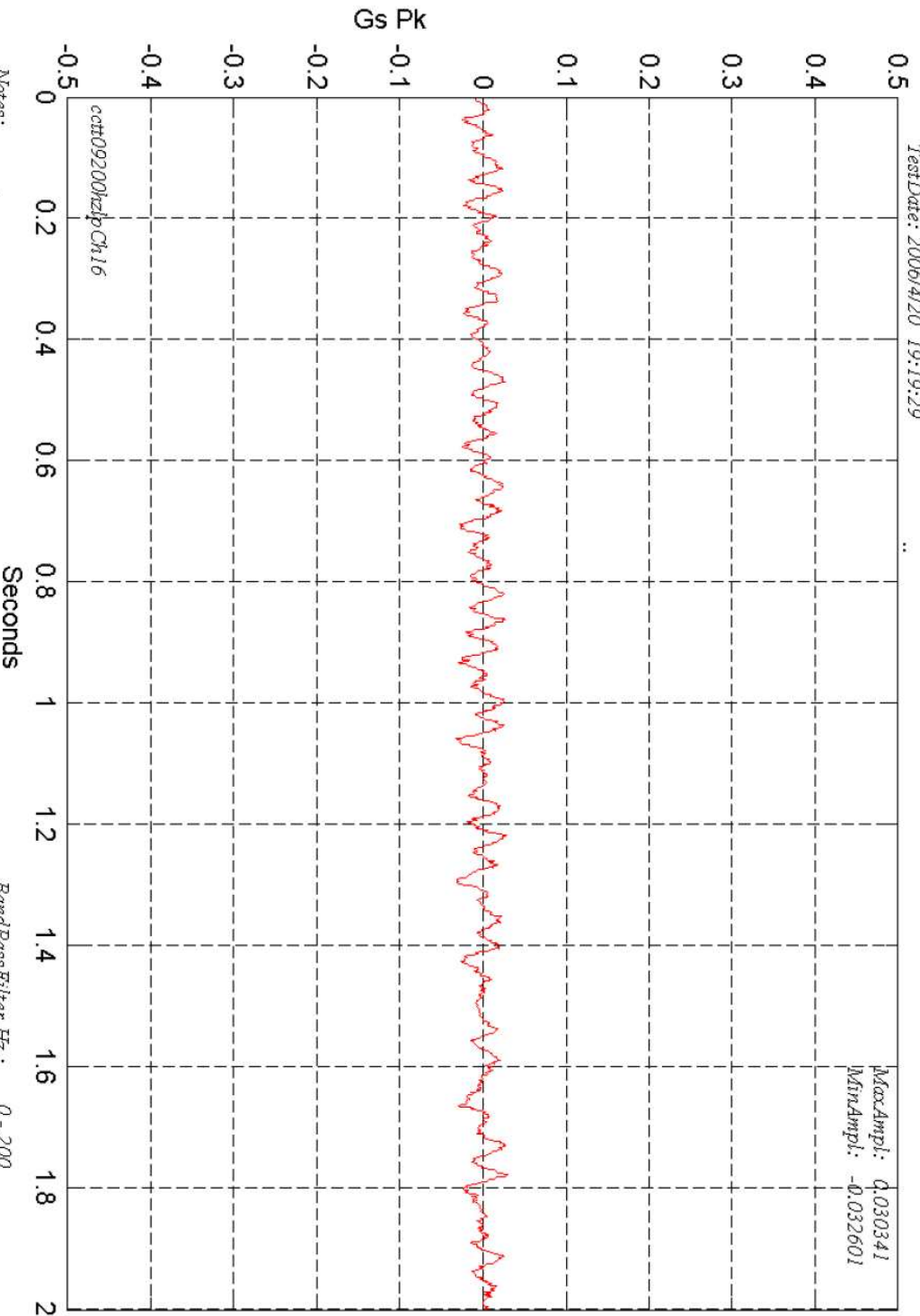
TestIdent : Weapon MK-19 Continuous

Time History Analysis

TestDate : 2006/4/20 19:19:29

TestCond : 200 Hz Bandwidth

Resp.Id : ProjMtgBaseX



Notes:

BandPassFilter Hz : 0 - 200
SampleRate S/Sec : 8192

Program: CCTT

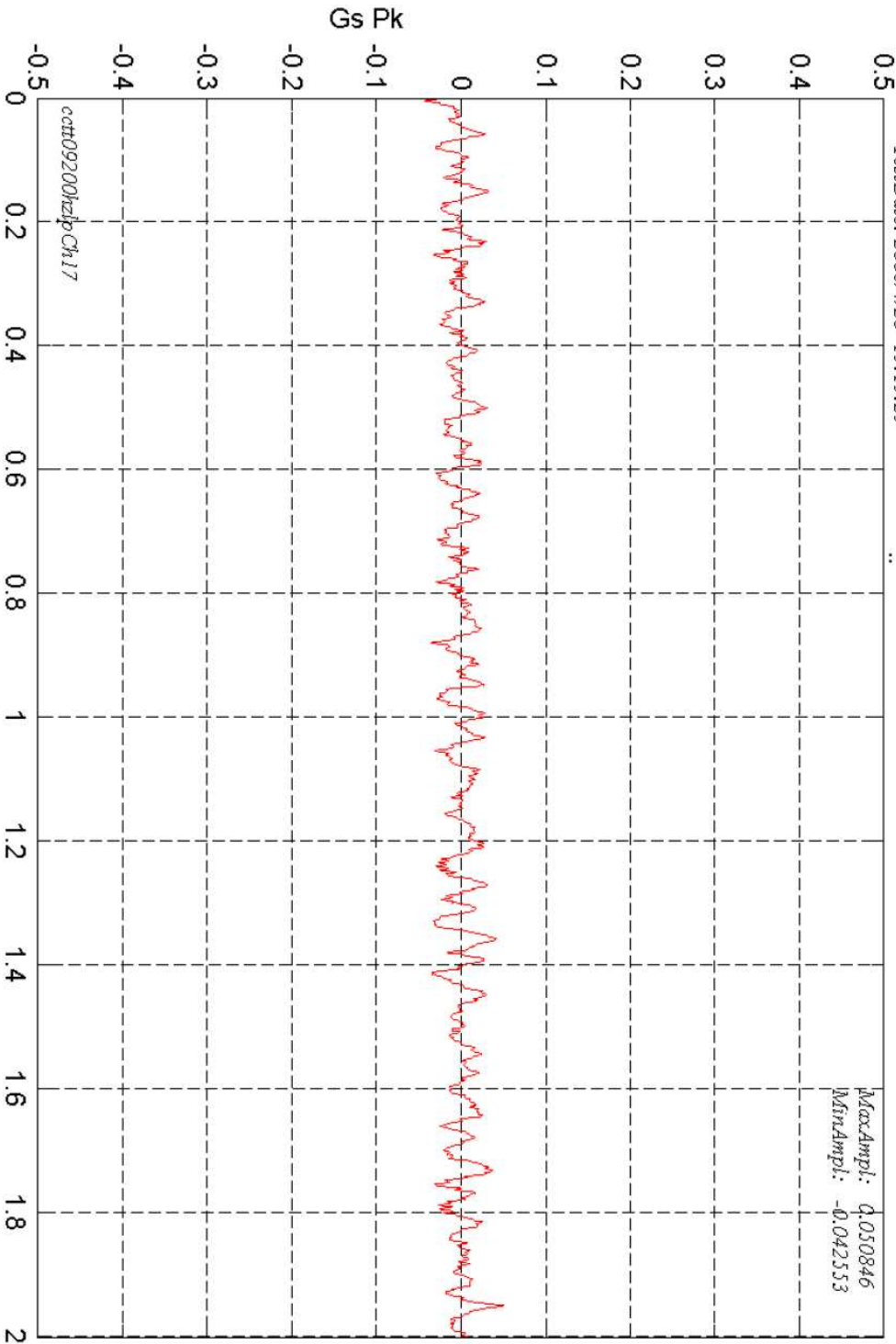
TestIdent: Weapon MK-19 Continuous

Time History Analysis

TestDate: 2006/4/20 19:19:29

TestCond: 200 Hz Bandwidth

Resp.Id: ProjMagBaseY



Notes: . . .

Notes:

BandPassFilter Hz: 0 - 200
SampleRate S/Sec: 8192

Program: CCTT

TestIdent: Weapon MK-19 Continuous

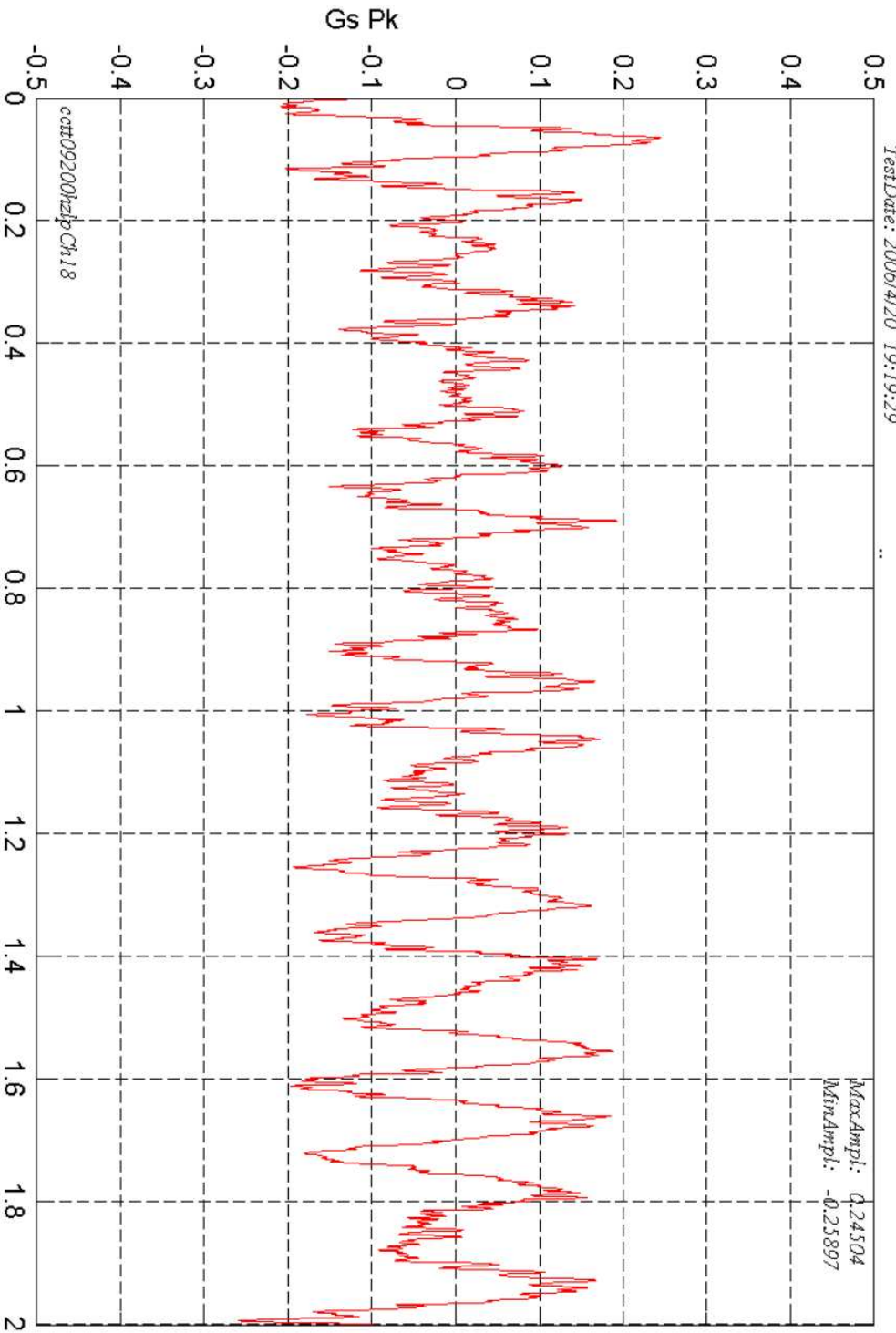
Time History Analysis

TestDate: 2006/4/20 19:19:29

TestCond: 200 Hz Bandwidth

Resp.Id: ProjMagBaseZ

MaxAmpl: 0.24504
MinAmpl: -0.25897



Notes:

Seconds

BandPassFilter Hz: 0 - 200

Notes:

SampleRate S/Sec: 8192

Program: CCTT

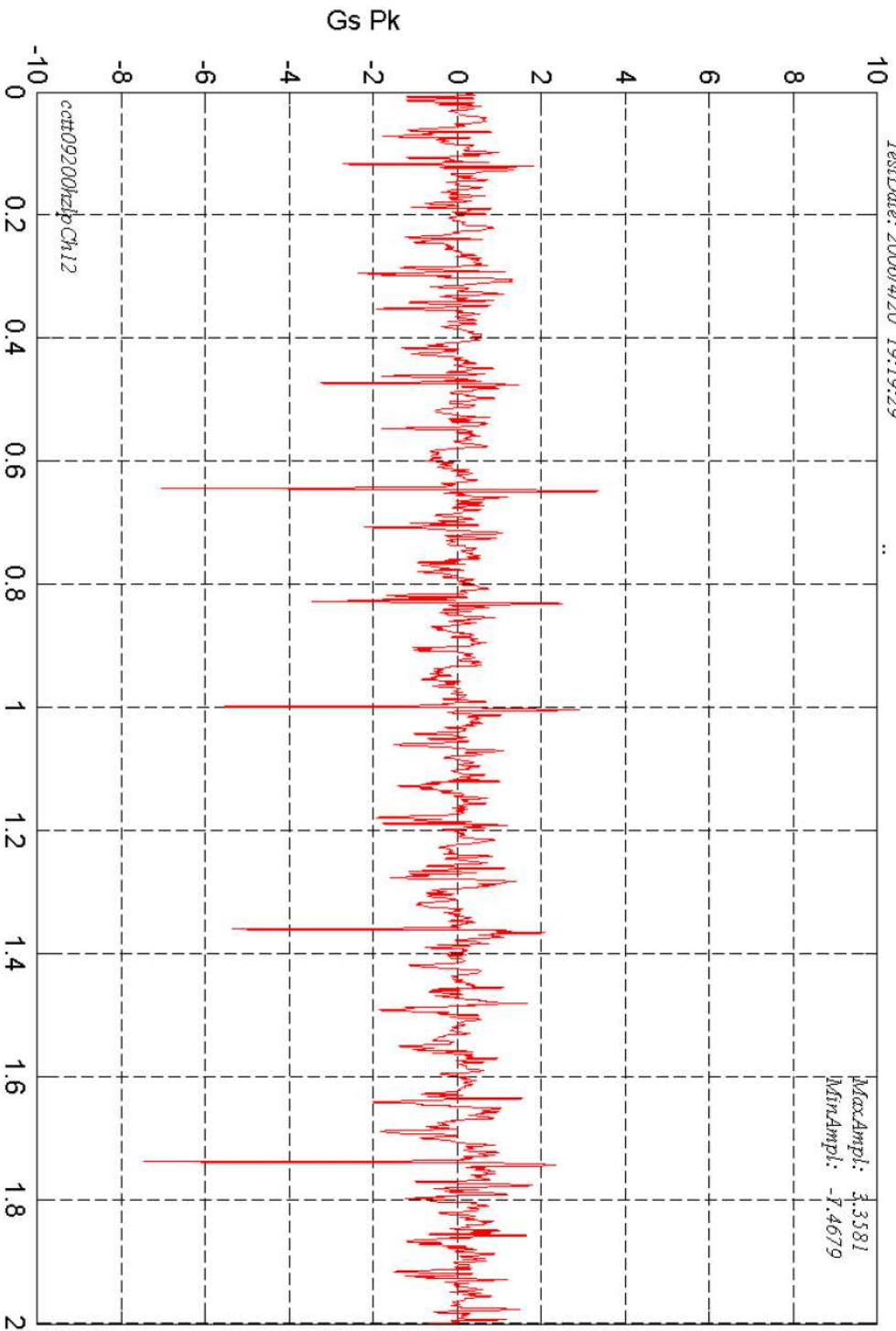
TestIdent: Weapon MK-19 Continuous

Time History Analysis

TestDate: 2006/4/20 19:19:29

TestCond: 200 Hz Bandwidth

Resp.Id: WeapPntBaseX



Notes: .

BandPassFilter Hz: 0 - 200
SampleRate S/Sec: 8192

Program: CCTI

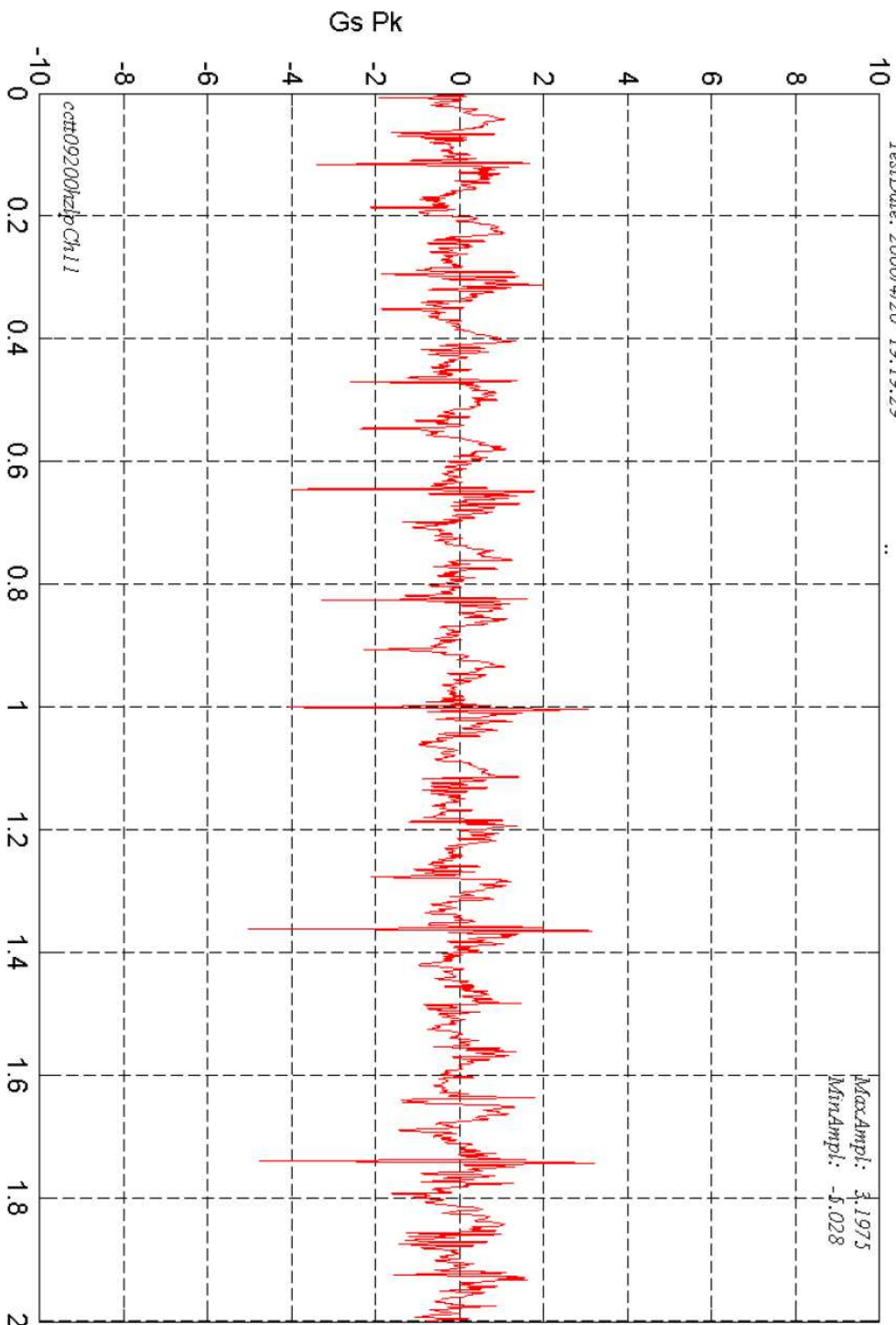
TestIdent: Weapon MK-19 Continuous

Time History Analysis

TestDate: 2006/4/20 19:19:29

TestCond : 200 Hz Bandwidth

Resp.Id: WeapPrintBaseY



Notes:

BandPassFilter Hz: 0 - 200
SampleRate S/Sec: 8192