

Conceptual Design

Group 1: Danfoss-Turbocor

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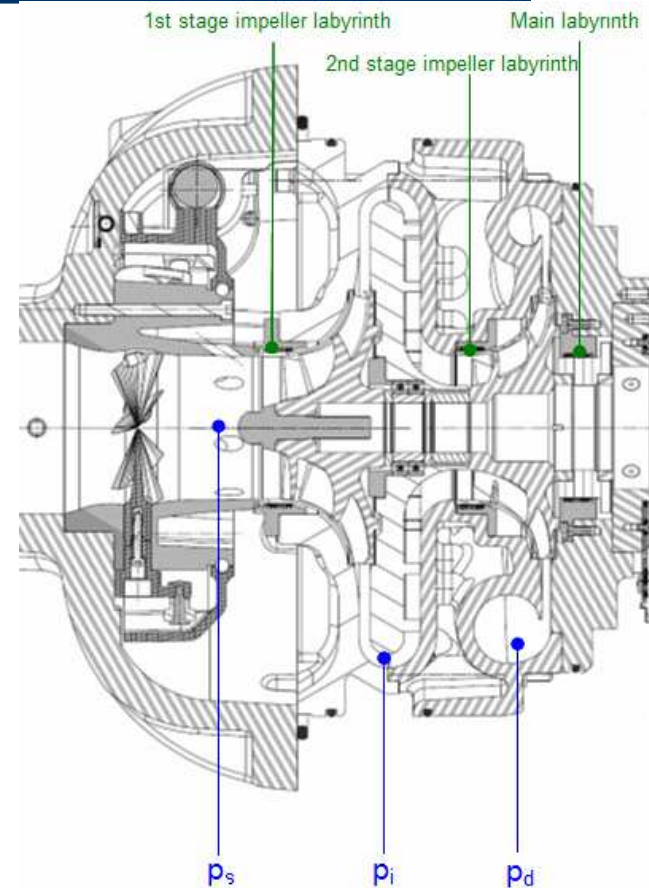
Danfoss-Turbocor: The Company

- Danfoss founded by Mads Clausen in 1933
- Manufactured automatic valves for refrigeration plants
- Turbocor started in Australia in 1993
- Headquarters is moved to Montreal in 1999
- Frictionless Compressors went on the market in 2002
- Denmark based Danfoss merges with Montreal based Turbocor in 2004
- Headquarters is moved to Tallahassee in 2007

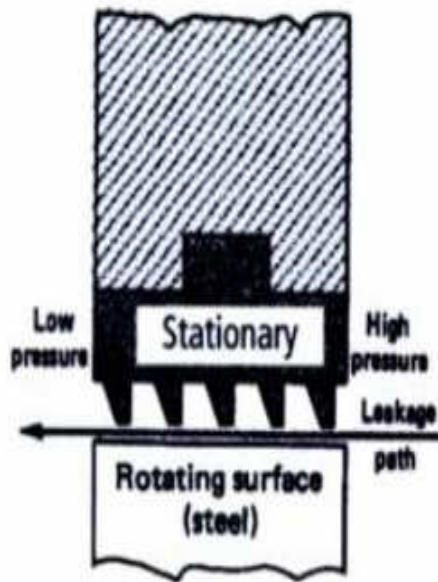


Problem Definition

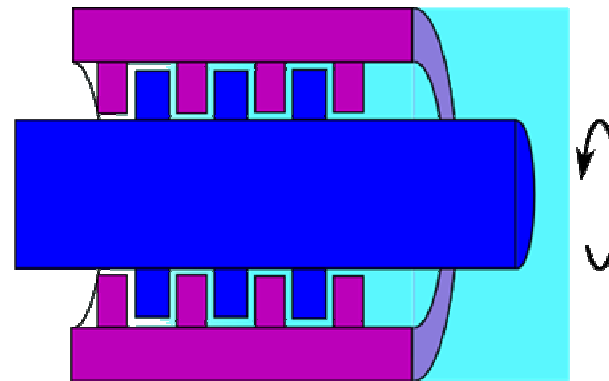
- Design and build a test rig that simulates conditions in a high speed centrifugal compressor
- The leakage flow through the seal must be measured to show which seal is superior
- Rig must allow for interchangeable seals for testing



Labyrinth Seals



- Mechanical seal that fits around a shaft to prevent leakage of fluid
- Provide non-contact sealing by controlling flow of fluid
- Threads create a “maze” to induce turbulence and block flow



Customer Needs

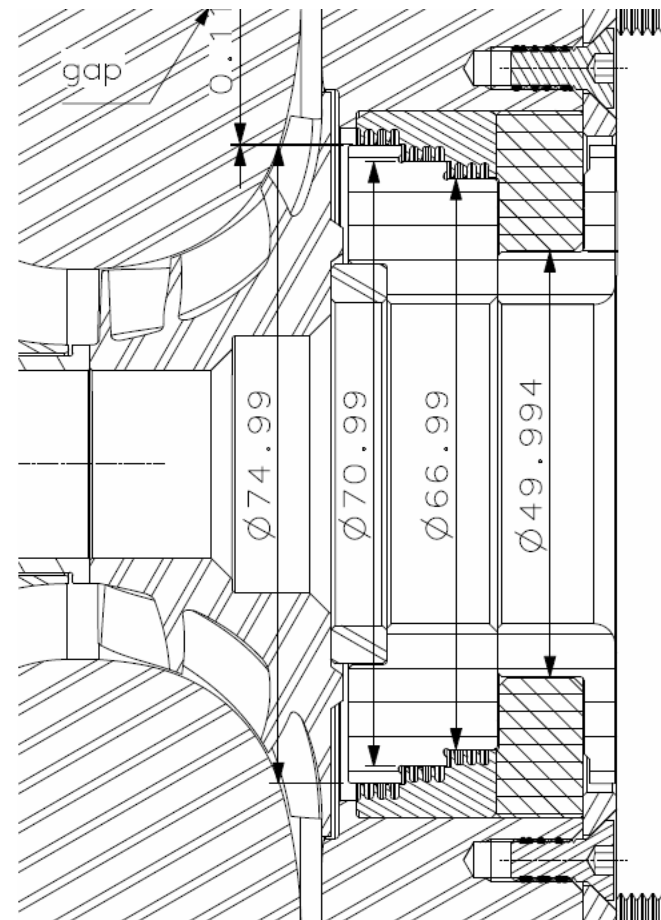
- Build a test rig that will quantitatively measure the leakage through a labyrinth seal
- Use air as the working fluid, and closely match the air with the properties of R-134a.
- A rotating shaft must be incorporated into the test rig
- The rig should be capable of adjusting to fit various seal designs and size

Product Specifications

- Set the Reynolds numbers of air and R134a equal to determine necessary conditions
 - The optimal inlet and outlet conditions are:
 - In: 2 °C & 171 kPa
 - Out: 55 °C & 1391 kPa
- The rig's shaft will operate at approximately 10,000 rpm
- The rig will be run with shaft concentricity varying at 100%, 90%, 80% and 70% centered
- 3 Labyrinth seal designs will be tested in various sizes
 - Impeller Labyrinths
 - Main Labyrinths
 - Interstage Labyrinths

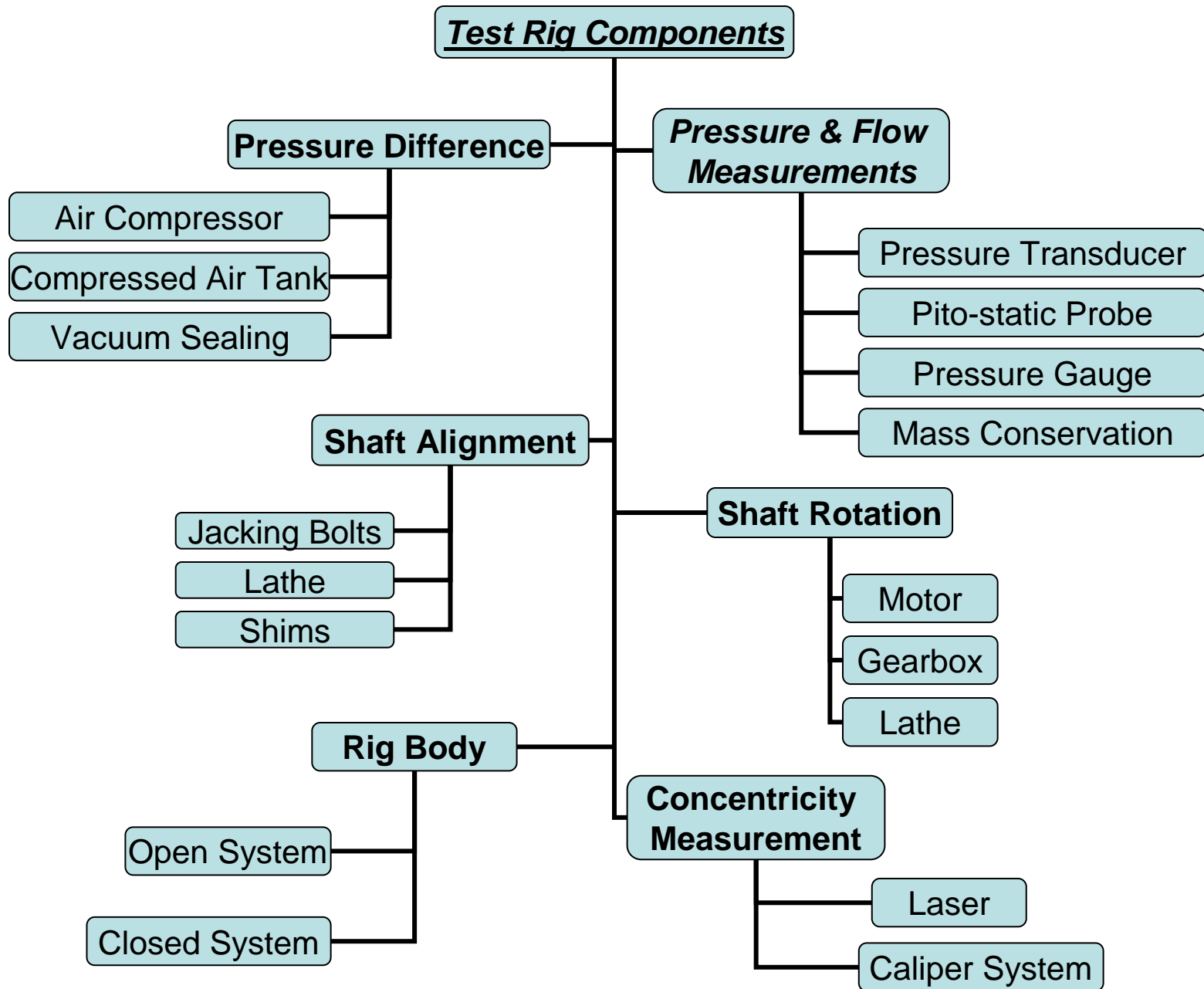
Some Analysis

- Iterative process to compare Reynolds number of air at various conditions to R134a
- Use “RefProp” to find conditions of air at varying Pressures and Temperatures
- Compare the Mach numbers of both fluids as a verification
- The tangential Velocity caused by the shaft is the dominant velocity



System Breakdown

- Shafting Considerations
 - Providing necessary rpm
 - Initial alignment
 - Variable concentricity
- Rig Body considerations
 - Maintaining a pressure gradient
 - Ability to test multiple designs and sizes of labyrinth seals
- Measurement Needs and Options
 - Air getting through the seal
 - Pitot-Static Probe
 - Pressure Transducers
 - Mass flow meter
 - Shaft Alignment
 - Calipers
 - Laser alignment



Decision Matrix

Parameters Idea	Manufacturability (5)	Cost (3)	Rig versatility (2)	Effectiveness (4.5)	Ease of use (3.5)	Repeatability (4)	Total (22)
Shaft Rotation							
Lathe	4	3	1	4.5	3.5	2	18
Motor	3	1	2	4	3.5	3.5	17
Seal Mounting							
chuck	2	1	2	2.5	3	4	14.5
Multiple custom plates	4	2.5	1	4	2.5	4	18
Pressure Difference							
Vacuum Sealing	2	1	1	2	1	3.5	10.5
Compressed air tank	4.5	3	2	3	3	2.5	18
Air compressor	4.5	2	2	4	3.5	4	20



Conceptual Design 1: Enclosed Seal





Design 1 Continued





Conceptual Design 2: Lathe Attachment





Design 2 Continued



Schedule: Past, Present, and Future

- **Completed:**
 - Initial Client Meeting
 - Needs Assessment
 - Product Specifications
 - Initial Brainstorming
- **Currently Open**
 - Conceptual Development
 - Design Selection
- **Future Endeavors**
 - 3D and 2D system diagrams
 - Material and parts selection
 - Parts Procurement

Any Questions?

