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TEAM 7
FAMU FSU ENGINEERING COLLEGE



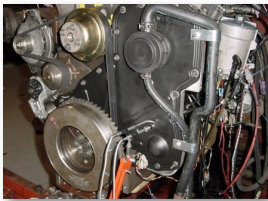
Cummins

CUMMINS ACTIVE NOISE CONTROL

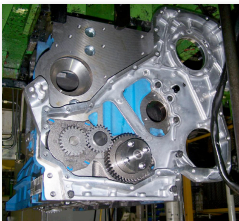
TEAM LEADER: JOSHUA HOGUE
ACTIVE CONTROLS: CHRIS SCHULTZ
3D MODELING: MARSHALL GOERG
DATA ANALYSIS: QUENNAN DAVIS
PASSIVE MATERIALS: MICHAEL PRIEBE

PROJECT DESCRIPTION

- Problem Statement:
 - Propagation of vibrations in valve or gear covers in mid-sized diesel engines result in radiated noise
 - Use active and passive materials to reduce noise
- Constraints:
 - Size and space limitations of engine
 - Broadband frequency control
 - Overall durability and long life



Gear Cover Mounted on Diesel Engine



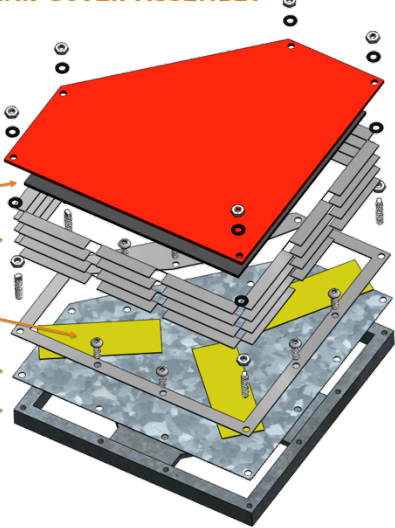
Gear Housing and Engine Components

- Project Specs:
 - Thin, cost-efficient, and lightweight vibration control devise utilizing both active and passive materials
 - Device should complement existing engine components
 - Cannot exceed 15mm from top of cover
 - Cannot surpass the existing perimeter of cover
 - Develop working model
 - Comparison of design vs. existing cover and dual-layer cover

SCALE MODEL OF GEAR COVER ASSEMBLY

- Scale Model Components:

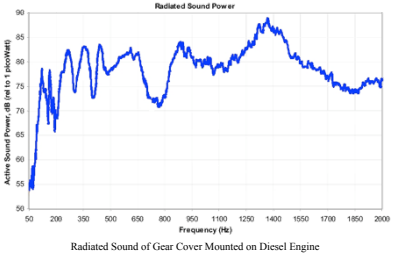
- Sound Barrier Panel
- Sound Absorbing Foam
- Vibra Block
- MFC Actuators
- Gear Cover
- Gear Housing



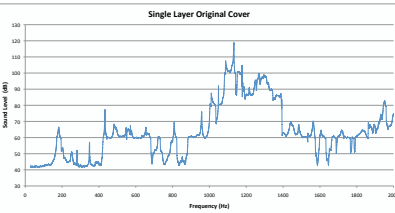
Shaker Table Testing Apparatus

- Shaker Table
 - Proven as accurate testing platform for substitution of diesel engine
 - Provides source of steady state perturbations

SOUND DATA



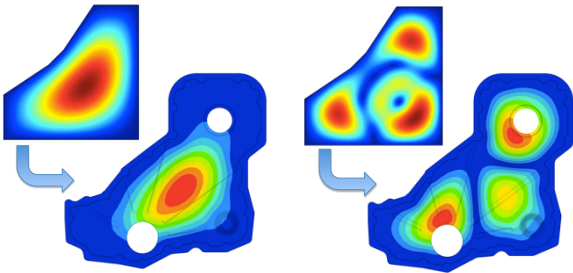
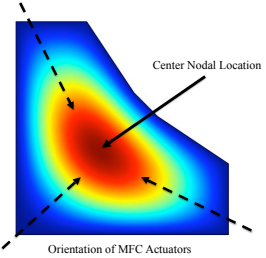
Radiated Sound of Gear Cover Mounted on Diesel Engine



Radiated Sound of Gear Cover Mounted on Shaker Table

MODAL ANALYSIS

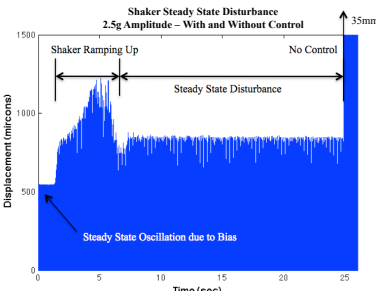
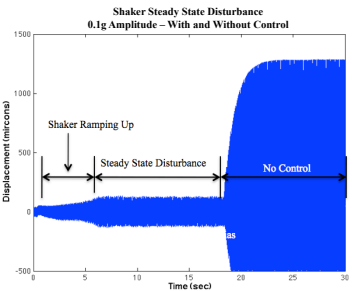
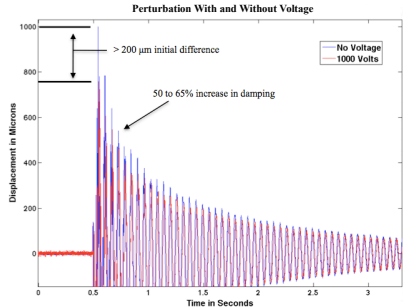
- FEA of Scale Model Design
 - Adaptation of gear cover shape for nodal location similarities
 - Nodal shape and location characterizes vibration propagation
 - Determine orientation of patch actuators (in direction of vibrations)



Similarities of FEA of Gear Covers

ACTIVE CONTROL

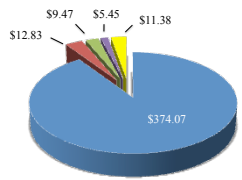
- Cantilever Beam Testing
 - Provides dynamic characteristics of materials
 - Implementation of lead lag filter / integral controller
- Results:
 - Increase in damping and stiffness
 - With control, 35 mm peak-to-peak reduced to 1.5 mm peak-to-peak displacement
- Proof of concept



RESULTS

Layering	Increase/Decrease	Sound Level	Frequency Range
Vibra Block	Decrease	3-5 dB	Broadband
MFCs (no voltage)	Decrease	20 dB	700-800 Hz
	Decrease	10-20 dB	1800-2000 Hz
	Decrease	Constant 5 dB	After 700 Hz
Control vs. No Control	Decrease	5 dB	@ 1150 & 1400
	Decrease	10 dB	@ 300, 450, 750
Sound Barrier	Decrease	10 dB	1250-1400 Hz
	Decrease	5 dB	1700-2000 Hz
Sound Absorbing Foam	Decrease	5 dB	Main Drop @ 500
	Decrease	5 dB	1400-2000 Hz

Model Cost (Per Component)



dB Drop (Per Component)

