

Team 2– Biaxial Tensile Tester

Team Members: Ben Hainsey
Eric Hebner
Nicole Walsh

Sponsor: Cummins, Inc. (Terry Shaw)

Graduate Consultant: Parker Harwood

Faculty Advisor: Dr. William Oates

Professional Aid: Bob Walsh and Scott Bole (MagLab)



Agenda

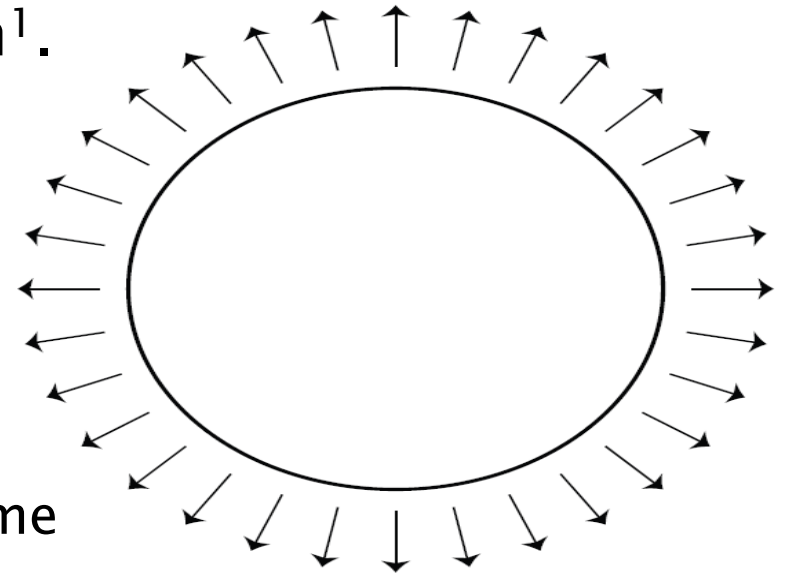
- ▶ Background/ Project Scope
- ▶ Specimen Geometry
- ▶ Material Testing
- ▶ Device Modifications
- ▶ Assembly
- ▶ Cable Testing
- ▶ Grip Testing
- ▶ Procurement
- ▶ Further Plans

Material Characterization

- In order to model materials, accurate predictions of properties are needed
 - **Uniaxial tension**
 - Easy to obtain with standard tensile test
 - **Pure shear**
 - Done with planar tension test
 - **Uniaxial Compression**
 - Inaccurate due to the friction between the load plates and the specimen
 - Causes a mixed state of compression, shear, and tensile strain¹

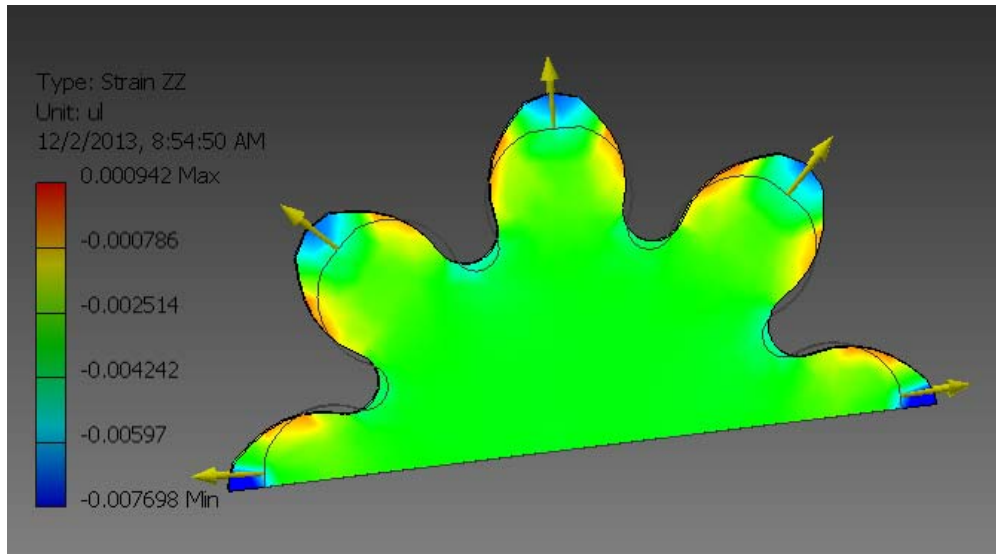
Why Equal Biaxial Tension?

- A biaxial tensile strain is equivalent to a uniaxial compressive strain¹.
- Mohr's Circle
 - Becomes a point circle
 - No shear forces are present³
- Poisson's Ratio nearly 0.5
 - Means a process of constant volume
 - $\gamma = -\frac{\epsilon_z}{\epsilon_x}$
- Free of the frictional effects²

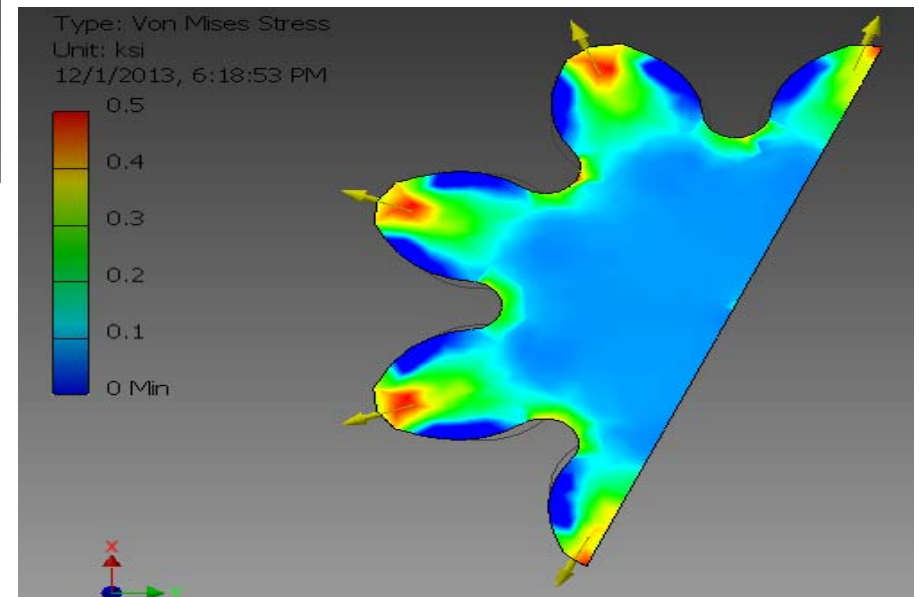


Ideal Equal Biaxial Stress State'

Final Specimen Geometry



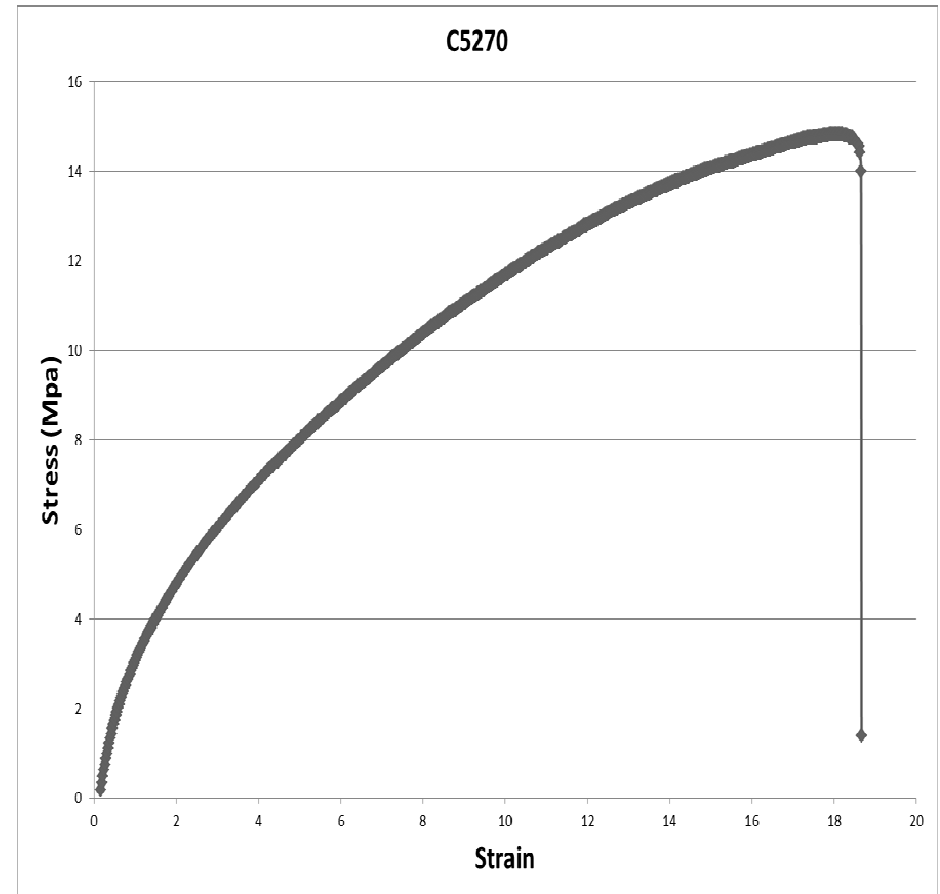
The strain profile in the ZZ plane after load is applied



The resulting Von Mises Stresses as force was applied radially

Material Testing

- ▶ The stiffest material, Gasket C5270, was tested in the traditional dog bone shape
- ▶ Maximum stress was found to be 14.8 MPa
- ▶ The max stress was used with a conservative cross-section of our sample geometry
- ▶ This gave an estimation of the maximum load to be used on our device of 1.054 kN



Device Changes

- ▶ Cut down diameter of all plates by 5 inches
 - Reduces weight by ~50 lb
- ▶ All other components remained the same except the carriers
 - Location is only part modified due to reduced baseplate
- ▶ Carriers had to be modified to fit cable that was decided upon

Grip Testing

- Mock grips constructed for use in MTS machine
- Testing for capability of grasping without slip
 - If slip occurs, surfacing metal to increase friction will be attempted first.
 - If slip still occurs, movement of clamping hole to increase grip surface area will be attempted
 - Testing for optimal torque



Cable Testing

- Initial testing of dyneema rope revealed knot slipped, eyebolt broke at 400lbf
 - New knots being investigated
 - Stronger eyebolt ordered
- Steel cabling has been acquired
- Mechanical fasteners being investigated
- Attachments for MTS machine fabricated



Budget

- Procured
 - Aluminum for plates, supports and carriers
 - Hardened linear rods and bearings
 - Threaded rods for attaching top and middle plates
 - Hardware for grips, supports, pulleys
- Remaining Items
 - Cables
 - Cable attachments
 - Machining
 - Strain gauges
- Remaining Budget
 - \$678.46

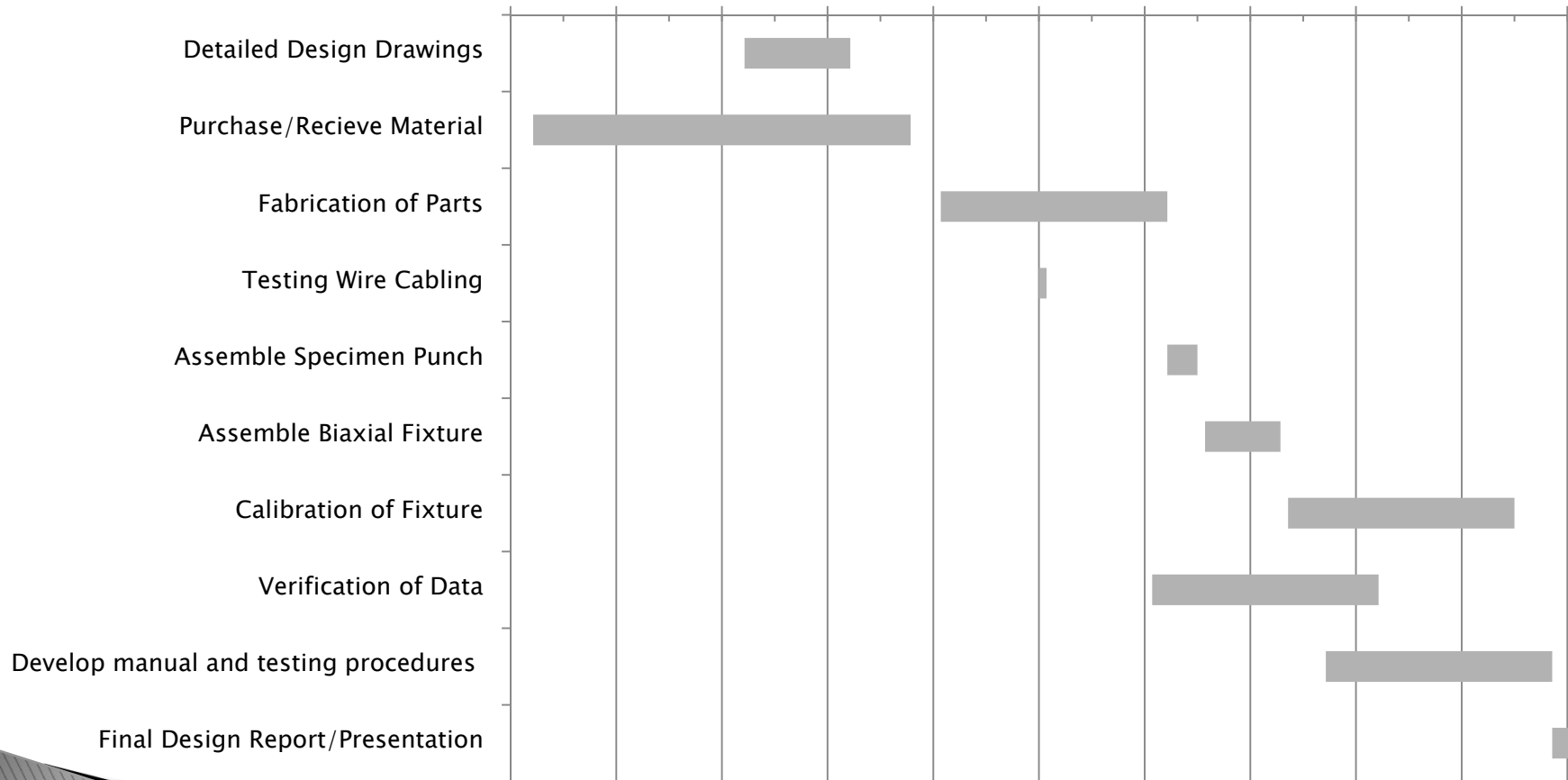
Future Work

- ▶ Cables to be tested
- ▶ Machining to start immediately
- ▶ Test remaining gasket materials in uniaxial tension
- ▶ Once device is built, begin proof testing
- ▶ Calibrate device to operate successfully
- ▶ Make changes if necessary
- ▶ Create working instructions for user

Gantt Chart

Spring 2014

6-Dec 20-Dec 3-Jan 17-Jan 31-Jan 14-Feb 28-Feb 14-Mar 28-Mar 11-Apr 25-Apr



References

1. <http://www.axelproducts.com/downloads/CompressionOrBiax.pdf>
2. Callister, W.D. (2007). *Material Science and Engineering, An Introduction; 7th ED.* York, PA: John Wiley & Sons, Inc.
3. Day, J. and Miller, K. (July 2000), Equibiaxial Stretching of Elastomeric Sheets, An Analytical Verification of Experimental Technique. *Equibiaxial Stretching, Rev 2. 1-8.*

Questions?
Comments?
Suggestions?