



Designing & Testing Thermal Management System for SiC PV Converter

Team 13

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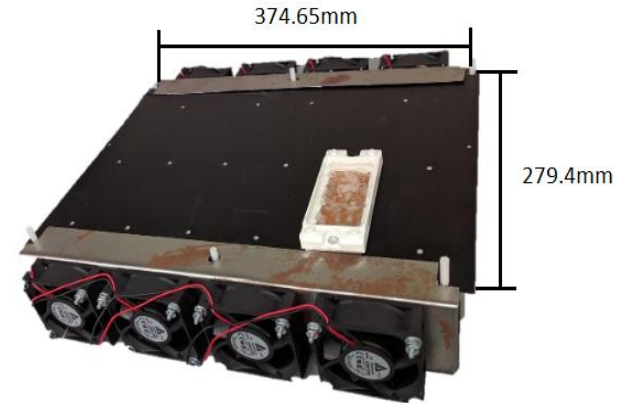
- PV Converters made with PE devices
 - Transform energy from solar array to usable
 - Generate heat
 - Cooled by heatsink/fans
- Typically made with Silicon
 - Cheap & abundant
 - Dominate the market
- SiC Power Electronic Module
 - Pro: Efficient
 - Con: Expensive
 - PowerAmerica goal: make SiC cost competitive



SiC Power Module

Project Overview

- **Background:** CAPS Next Generation PV Converter has one of the highest power densities (Power/weight)
 - Further increase power density
- **Problem:** Heatsink used for this PV Converter was oversized
 - Remains cool during operation
 - Too heavy
- **Solution:** Provide an optimal heatsink design
 - Decrease weight of system, increase power density
- **Approach:** 3 methods to verify design
 - Simulation, Calculations, & Experimentation



Next Gen PV Converter

- Plate Fin Heatsink
- 8 Power Modules & Fans
- 375 mm x 280 mm x 80 mm
- 6.5 kg

Bi-Modular Design for Heatsink

Heatsink Design	Plate Fin	Pin Fin
Size	127mm x 127mm x 69.2mm	113.7mm x 113.7mm x 17.8mm
Weight (including fans)	0.954 kg	0.553 kg
Fan Orientation	Lateral	Axial

Heat Source Emulator

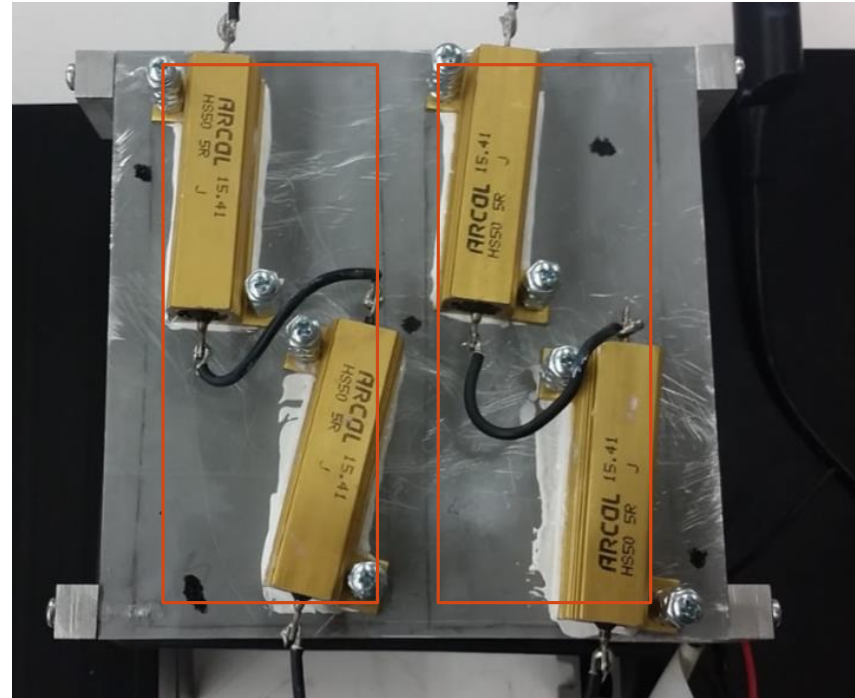
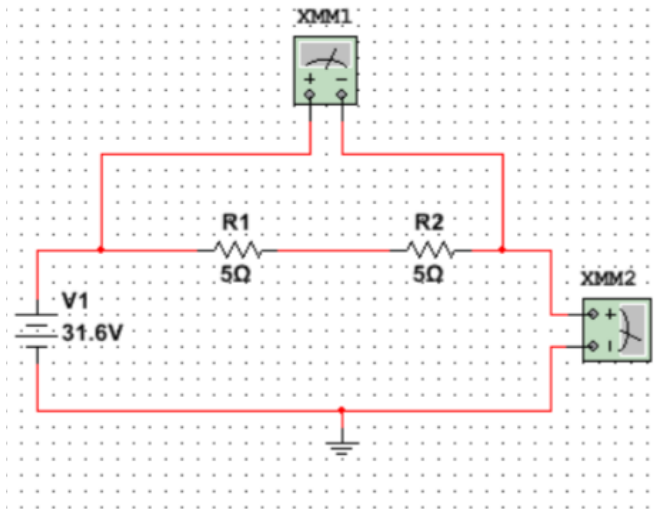


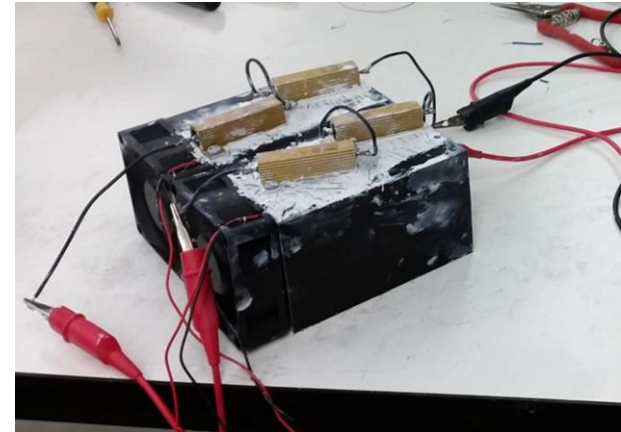
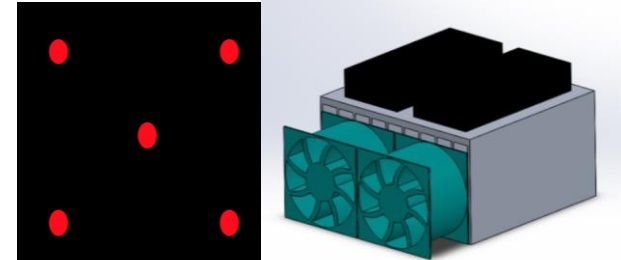
Plate Fin Testing

- With copper plate

Power Diss. /Emulator (W)	Total Power Diss. (W)	Average Temp. (°C)
0	0	23.36
15	30	28.04
30	60	30.36
45	90	33.22
60	120	36.4
75	150	42.08
90	180	42.5

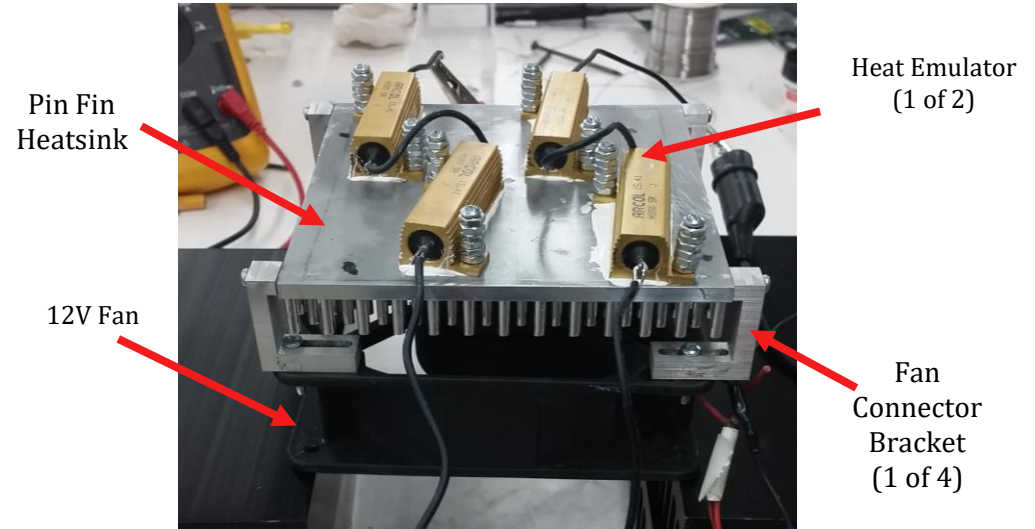
- Without copper plate

Power Diss. /Emulator (W)	Total Power Diss. (W)	Average Temp. (°C)
0	0	23.9
15	30	28.16
30	60	29.22
45	90	34.6
60	120	37.2
75	150	39.6
90	180	41.4

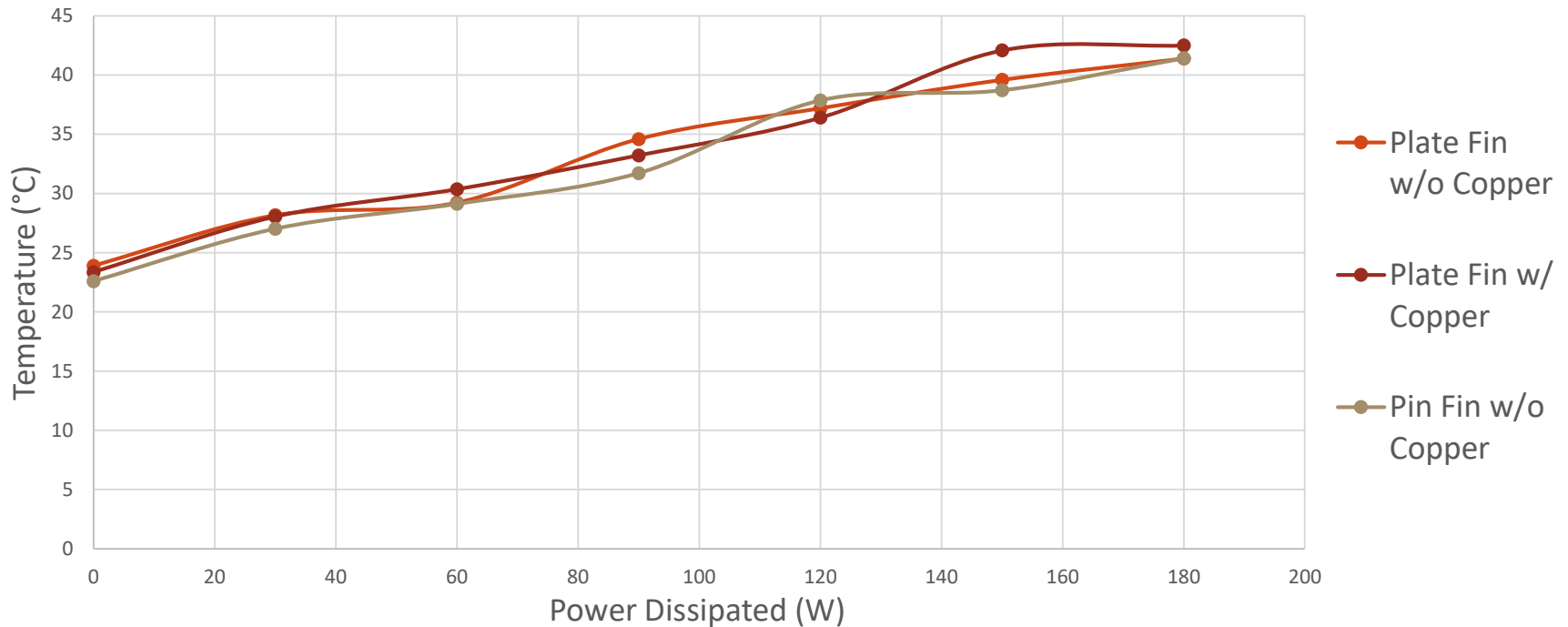


Pin Fin Testing

Power Dissipated per Emulator (W)	Total Power Dissipated (W)	Average Temperature (°C)
0	0	22.6
15	30	27.02
30	60	29.12
45	90	31.72
60	120	37.86
75	150	38.72
90	180	41.42



Experimentation Results



Project Approach

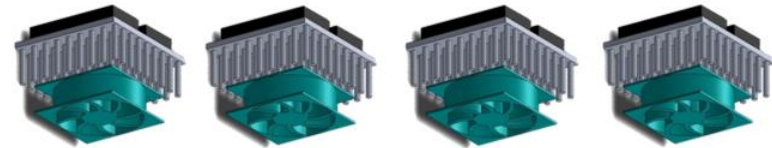
Original CAPS Heatsink

- Plate Fin Heatsink
- 8 Power Modules & Fans
- 375 mm x 280 mm x 80 mm
- 6.5 kg



Solution: 4 Pin Fin Heatsinks

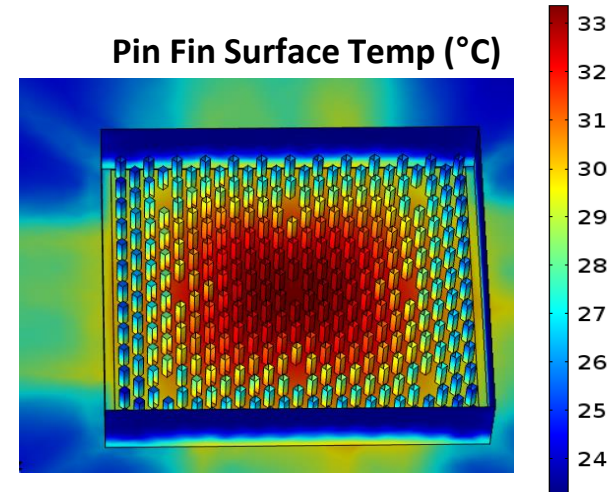
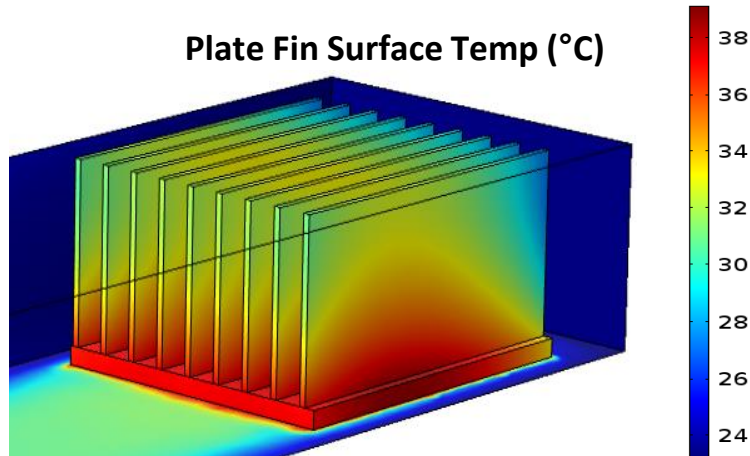
- Pin Fin Heatsink
- 2 Power Modules / Heatsink
- 1 Fan / Heatsink
- 115 mm x 115 mm x 14.7 mm
- ≤ 0.56 kg per heatsink (2.2 kg total)



Comparison of Plate vs Pin

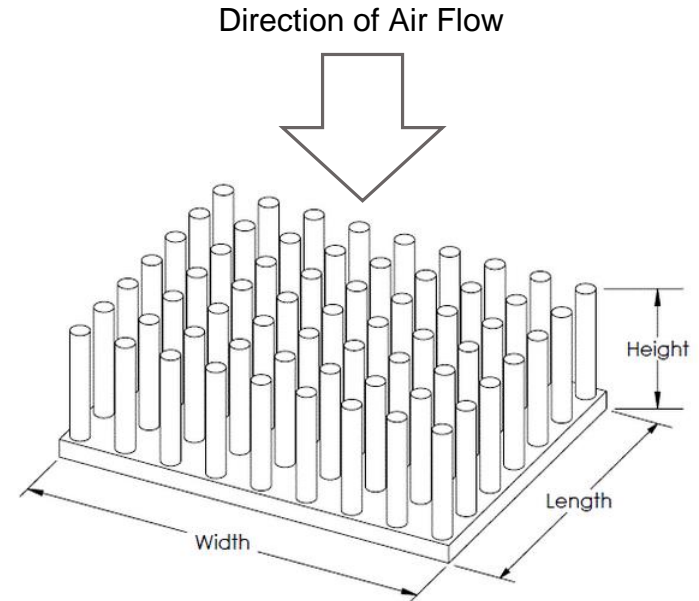
Pin fin heatsink design has been chosen for optimization over the plate fin heatsink

- Weight reduction is more significant for pin fin
- Thermal performance of pin fin virtually equal to that of plate fin
- Pin fin equations provide less error than plate fin equations



Theoretical Analysis

- Convective Thermal Resistance
 - Reynolds Number
 - Nusselt Number
 - Fin Efficiency
- Conductive Thermal Resistance
 - Geometry
 - Thermal conductivity
- Weight



Heatsink Optimization

Optimizing by varying heatsink geometry and fan speed.

Input Values	Output Values	Constant Values
<ul style="list-style-type: none">• Length of pins (5mm-40mm)• Diameter of pins (2mm-5mm)• Pin Spacing• Number of Pins• Fan Speed	<ul style="list-style-type: none">• Total Weight• Thermal Resistance	<ul style="list-style-type: none">• Base Size• Base Thickness

Heatsink Optimization

Goals

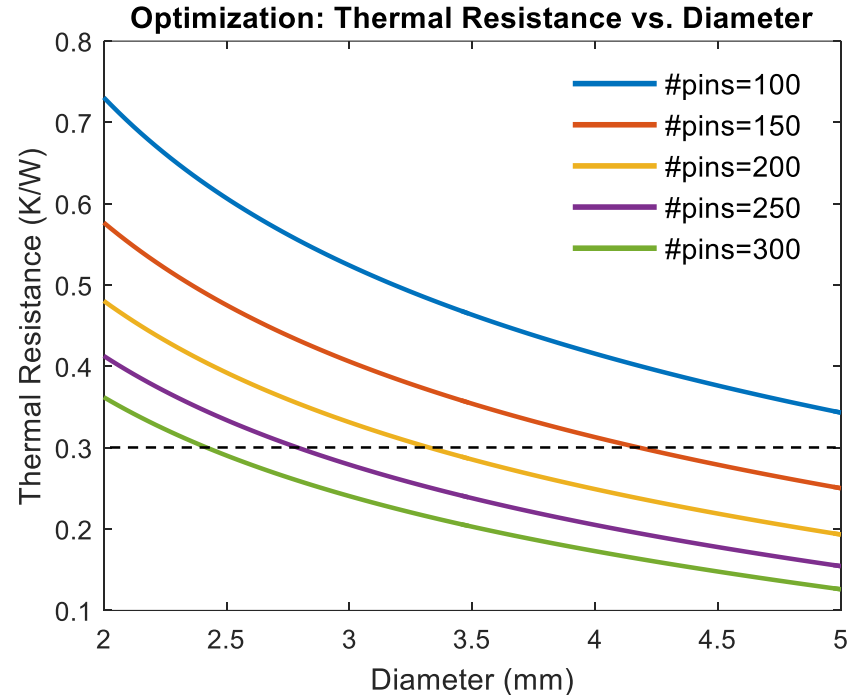
- Minimize heatsink weight (< 0.254 kg)
- Obtain thermal resistance of 0.3 K/W or less
- Baseplate temperature should be in the range 30 - 60°C

Assumptions

- Constant baseplate size: 115 mm x 115 mm x 4.7 mm
- Power loss/Heat source: 105.2 W

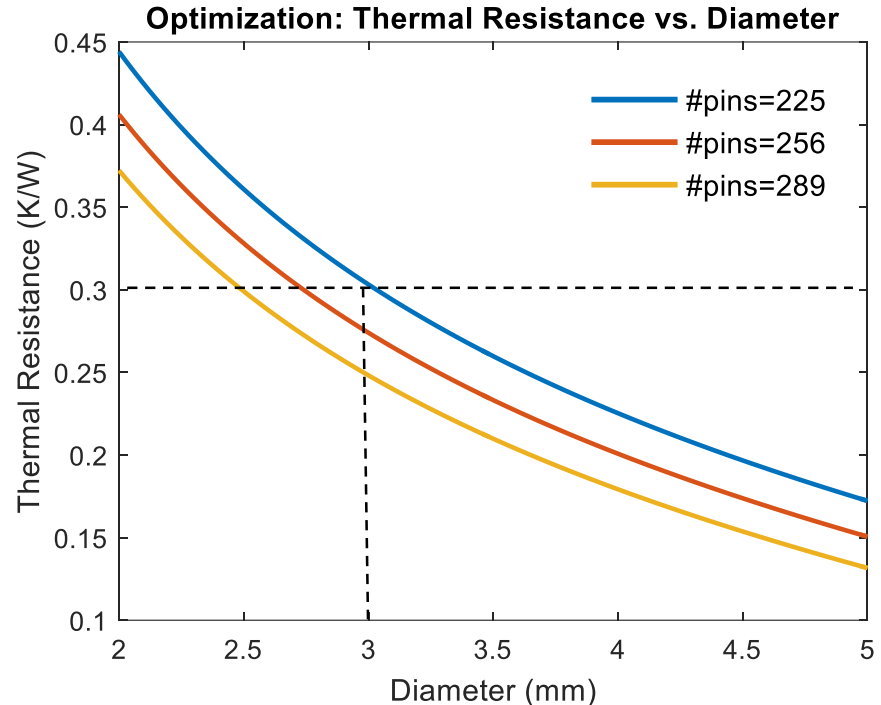
Optimization: Diameter

- Varying diameter size: 2-5 mm
- Each line represents different number of pins from 100-300
- To obtain necessary thermal resistance, #pins > 200
- Minimum diameter **3.0 mm** based on results



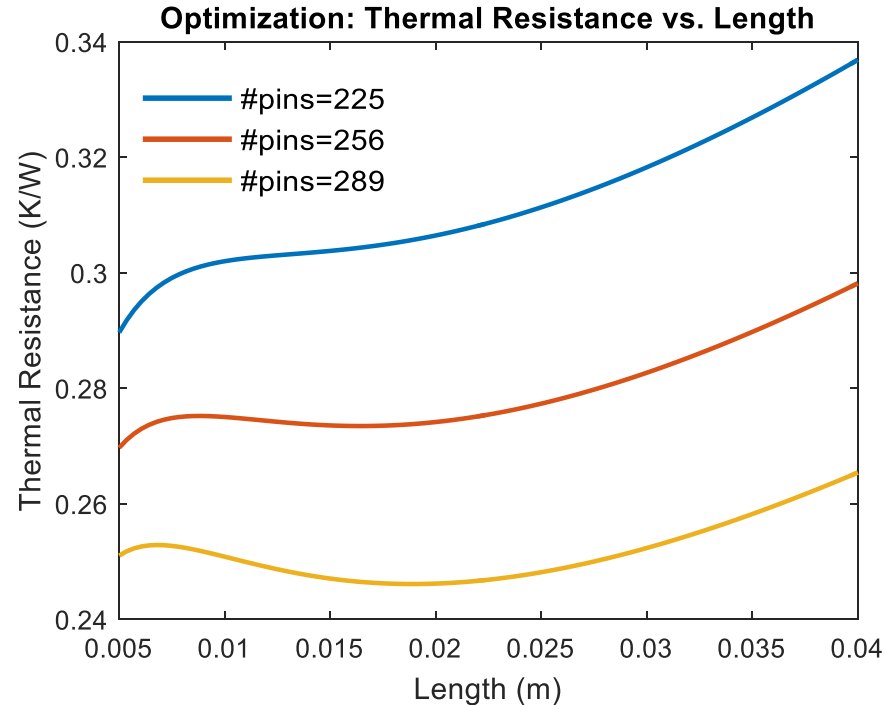
Optimization: # of Pins

- A uniform number of pins in each direction is needed to obtain even spacing
- Looking in the range of 200-300 total pins:
 - **15x15 = 225 pins**
 - **16x16 = 256 pins**
 - **17x17 = 289 pins**
- Acceptable thermal resistance is obtained at $D=3.0$ mm for each pin variation



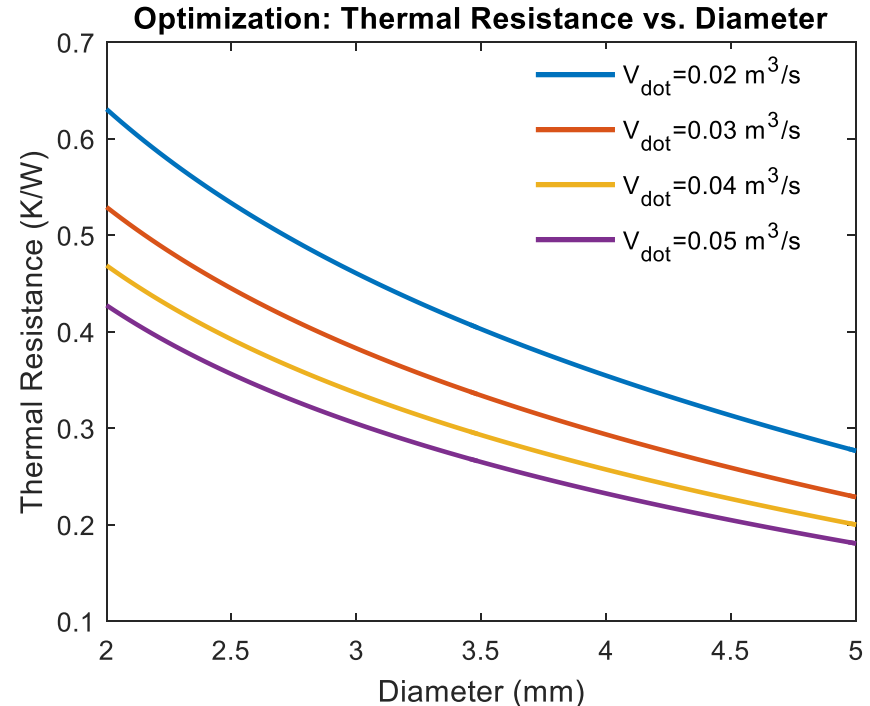
Optimization: Length

- Varying pin length: 5-40 mm
- Change in length affects thermal resistance far less than change in diameter
- To minimize weight and have dependable thermal resistance, **10 mm** length was chosen



Optimization: Fan Speed

- Varying fan flow rate:
0.02-0.05 m³/s
- Increasing the speed, lowers thermal resistance
- Fan must operate at least at **0.04 m³/s** to obtain desired thermal resistance



Fan Selection

- Searched for lowest weight fan that met sizing, flow rate, and voltage requirements
- New fan nearly **half the weight** of old fan

	Old Fan	New Fan
Weight (g)	300	157
Size (mm ³)	120 x 120 x 38	120 x 120 x 25
Flow Rate (m ³ /s)	0.0505	0.051
Voltage (VDC)	12	12

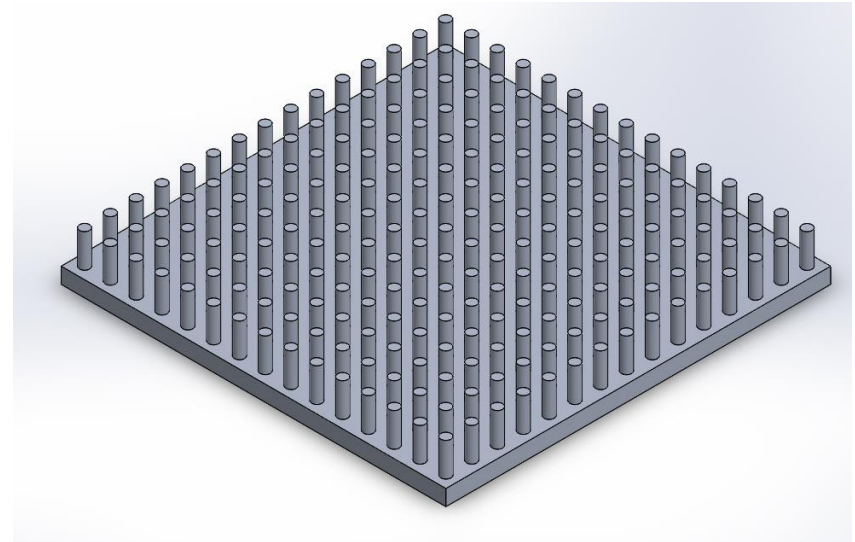


Optimized Design

- 15 x 15 array of evenly spaced pins (225 total)
- Diameter = 3.0 mm
- Length = 10.0 mm
- **Weight = 211 g**

Weight reduction including fans:

- 34% < Manufacturer pin fin
- 77% < CAPS original heatsink

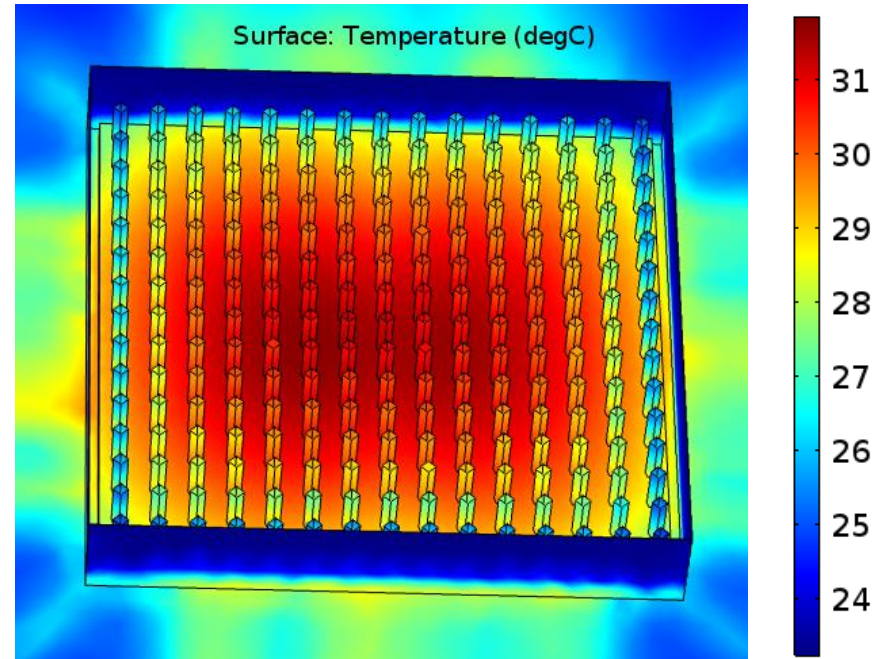


Optimized Pin Fin

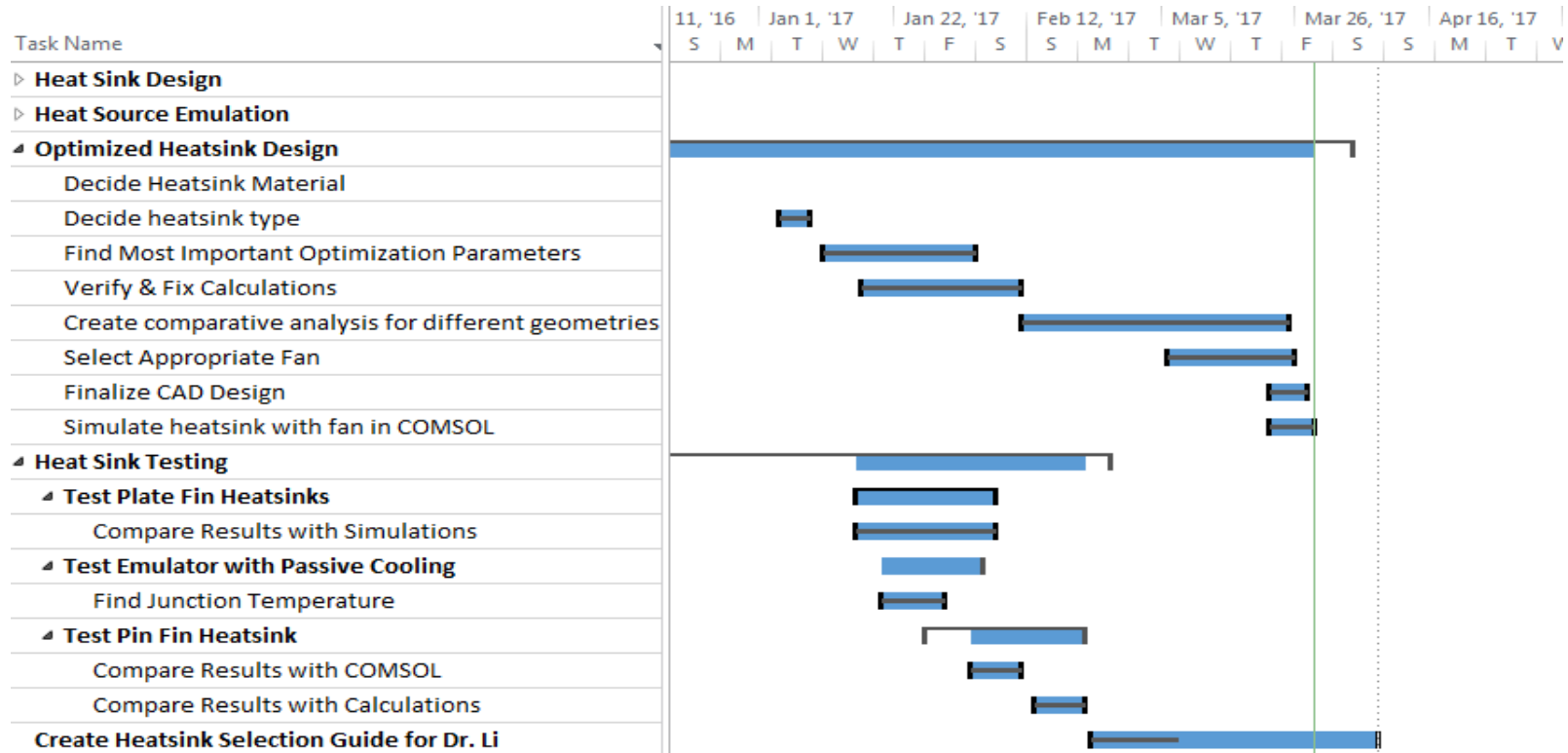
COMSOL Verification

- Max Surface Temperature = 31.6°C
- Yields thermal resistance ≈ 0.3004 K/W

$$R_{total} = \frac{\Delta T}{\dot{Q}}$$



Gantt Chart



Summary

Accomplishments

- Experimentally tested both pin fin and plate fin designs
- Conducted finite element analysis of pin & plate fin with COMSOL
- Developed analytical equations for pin fin heatsink
- Optimized the pin fin heatsink design

Future Plans

- Provide Dr. Li with a Heatsink Selection Guide

References

Çengel, Yunus A., Mehmet Kanoglu, John M. Cimbala, and Robert H. Turner. *Fundamentals of Thermal-fluid Sciences*. Singapore: McGraw-Hill Education, 2017. Print.

"Cold Forging Technology and Pin Fin Heat Sinks." *My Heat Sinks*. Web. 23 Feb. 2017.

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Questions?