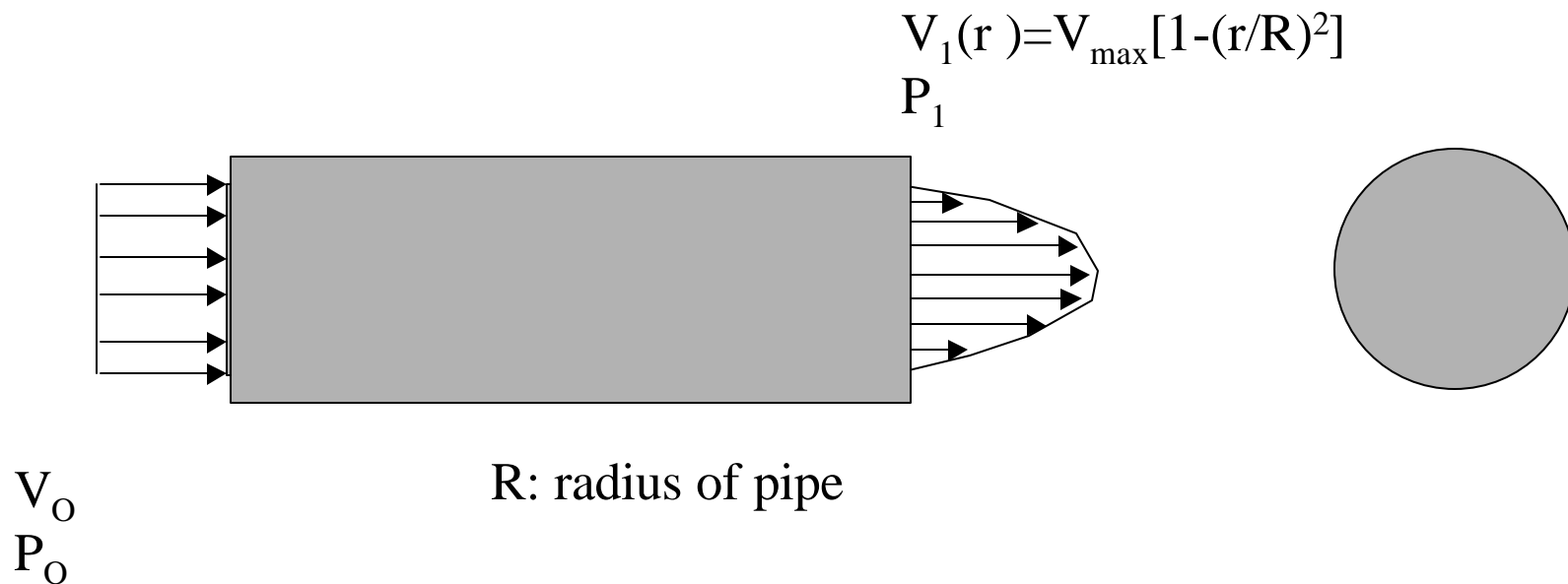


Pipe Flow Example

Water flows steadily into the circular pipe with a uniform inlet velocity profile as shown. Due to the presence of viscosity, the velocity immediately adjacent to the inner pipe wall becomes zero and this phenomenon is called the no-slip boundary condition. It is found out that the velocity distribution reaches a parabolic profile at a distance downstream of the entrance and can be represented as $V_1(r) = V_{\max}[1 - (r/R)^2]$, where V_{\max} is the velocity at the center of the pipe and r is the radial distance measured away from the center axis. Use the mass conservation equation, determine V_{\max} .



Mass Conservation

Mass conservation: $\frac{\partial}{\partial t} \int_{CV} \rho dV + \int_{CS} \rho \vec{V} \cdot d\vec{A} = 0$

$\int_{IN} \rho \vec{V} \cdot d\vec{A} + \int_{OUT} \rho \vec{V} \cdot d\vec{A} = 0$ 0, steady state

$-\rho V_o A_o + \int_{OUT} \rho V_1 dA_1 = 0$, since \vec{V} & $d\vec{A}$ are in the same direction

$V_o A_o = \int_{r=0}^{r=R} V_1(r) (2\rho r dr)$, since $dA_1 = d(\rho r^2) = 2\rho r dr$

$$V_o (\rho R^2) = \int_{r=0}^{r=R} V_{\max} \left(1 - \frac{r^2}{R^2} \right) (2\rho r dr) = 2\rho V_{\max} \int_0^R \left(r - \frac{r^3}{R^2} \right) dr$$

$$= 2\rho V_{\max} \left(\frac{R^2}{4} \right) = \frac{1}{2} \rho R^2 V_{\max}$$

Therefore, $V_{\max} = 2V_o$

Momentum Conservation

Assume there is no significant forces acting between the pipe and the fluid except the pressure forces normal to the pipe inlet (P_o) and the section 1 (P_1). Use linear momentum conservation equation, estimate the pressure difference between these two sections in order to accelerate the velocity profile inside the pipe from the inlet to a parabolic profile at section 1. Assume the pressure is uniform both at the inlet and section 1.

$$\text{Momentum conservation: } \vec{F}_S + \vec{F}_B = \frac{\partial}{\partial t} \int_{CV} \rho \vec{V} dV + \int_{CS} \rho \vec{V} (\vec{V} \cdot d\vec{A})$$

$$P_o A_o - P_1 A_1 + R_x = \int_{\text{inlet}} \rho \vec{V} (\vec{V} \cdot d\vec{A}) + \int_{\text{section 1}} \rho \vec{V} (\vec{V} \cdot d\vec{A})$$

$$(P_o - P_1) A_o = -\rho V_o^2 A_o + \int_0^R \rho V_1^2 d(\rho r^2)$$

$$= -\rho V_o^2 A_o + \int_0^R \rho V_{\max}^2 \left(1 - \frac{r^2}{R^2} \right) (2\rho r dr)$$

Momentum Conservation (cont.)

$$\begin{aligned}(P_o - P_1)A_o &= -\mathbf{r}V_o^2 A_o + \int_0^R \mathbf{r}V_{\max}^2 \left(1 - \frac{r^2}{R^2}\right)^2 (2\mathbf{p}rdr) \\&= -\mathbf{r}V_o^2 A_o + 2\mathbf{p}\mathbf{r}V_{\max}^2 \int_0^R \left(1 - \frac{2r^2}{R^2} + \frac{r^4}{R^4}\right) (rdr) \\&= -\mathbf{r}V_o^2 A_o + 2\mathbf{p}\mathbf{r}V_{\max}^2 \int_0^R \left(r - \frac{2r^3}{R^2} + \frac{r^5}{R^4}\right) dr \\&= -\mathbf{r}V_o^2 A_o + 2\mathbf{p}\mathbf{r}V_{\max}^2 \left[\frac{r^2}{2} - \frac{2r^4}{4R^2} + \frac{r^6}{6R^4} \right]_{r=0}^{r=R} \\&= -\mathbf{r}V_o^2 A_o + 2\mathbf{p}\mathbf{r}V_{\max}^2 \left(\frac{R^2}{6} \right) = -\mathbf{r}V_o^2 A_o + \left(\frac{4}{3} \right) \mathbf{r}V_o^2 A_o \\P_o - P_1 &= \left(\frac{1}{3} \right) \mathbf{r}V_o^2\end{aligned}$$