

Poiseuille flow



- Assumptions
- (1) Incompressible
 - (2) Newtonian
 - (3) Laminar, steady velocity field
 - (4) velocity is independent of \$t\$
 - (5) velocity is independent of \$z\$
 - (6) nonsingular at \$r=0\$

Continuity

$$\frac{1}{r} \frac{\partial}{\partial t} (r v_r) + \frac{1}{r} \frac{\partial}{\partial r} (r v_r) + \frac{1}{r} \frac{\partial}{\partial z} (r v_z) = 0$$

Using assumptions (4) and (5), the equation simplifies to:

$$\frac{\partial v_r}{\partial r} = 0 \rightarrow r v_r = C(r, z, t)$$

Boundary condition at \$r=R\$: \$v_r = \frac{C}{R} = 0 \rightarrow C=0\$

\$\rightarrow\$ $v_r = 0$ everywhere (7)

Students
 $v_r = 0$ at \$r=R\$
 If so, what assumptions are B.C. are not needed

\$r\$-momentum

$$\rho \left(\frac{\partial v_r}{\partial t} + v_r \frac{\partial v_r}{\partial r} + \frac{v_r v_\theta}{r} \frac{\partial v_r}{\partial \theta} + \frac{v_r v_z}{r} \frac{\partial v_r}{\partial z} + v_z \frac{\partial v_r}{\partial z} \right) = -\frac{\partial p}{\partial r} + \mu \left[\frac{\partial}{\partial r} \left(\frac{1}{r} \frac{\partial (r v_r)}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 v_r}{\partial \theta^2} + \frac{\partial^2 v_r}{\partial z^2} + \frac{2}{r} \frac{\partial v_r}{\partial r} \right]$$

Using assumptions (1), (2), (3), (4), and (5), the equation simplifies to:

$$\frac{\partial p}{\partial r} = \mu r \frac{\partial}{\partial r} \left(\frac{1}{r} \frac{\partial v_r}{\partial r} \right) = f(r, z, t)$$

~~\$p = f(r) + C(r, z, t) = p(r)\$~~

\$p = p(r, z, t)\$

\$p(r, \theta, z, t) = f(r) + p_0(r, z, t)\$
 so \$p(r, 2\pi, z, t) = p(r, 0, z, t) + f(r) 2\pi\$

Students
 what can \$f\$ say about \$p(r, \theta, z, t)\$
 can it depend on \$\theta\$?

But $p(r, 2\pi, z, t) = p(r, 0, z, t)$ (52)

so $f(r)$ must be $\phi \rightarrow p = p_0(r, z, t)$ (8)

also $f(r) = \mu r \frac{\partial}{\partial r} \left(\frac{1}{r} \frac{\partial v_{\theta}}{\partial r} \right) = 0 \rightarrow \frac{1}{r} \frac{\partial v_{\theta}}{\partial r} = C$

$\rightarrow \frac{\partial v_{\theta}}{\partial r} = Cr \rightarrow r v_{\theta} = \frac{1}{2} Cr^2 + D$ $\rightarrow D$ must be ϕ or $v_{\theta} = \infty$ at $r=0$
 $v_{\theta} = \frac{1}{2} Cr + \frac{D}{r}$

$v_{\theta} = \frac{1}{2} Cr$, but at $r=R$ $v_{\theta}|_R = \frac{1}{2} CR = 0 \rightarrow C=0$

$\rightarrow v_{\theta} \equiv 0$ (9)

r - momentum

$\rho \left(\frac{\partial v_r}{\partial t} + v_r \frac{\partial v_r}{\partial r} + \frac{v_{\theta}}{r} \frac{\partial v_r}{\partial \theta} + v_z \frac{\partial v_r}{\partial z} \right) = \mu \left[\frac{\partial}{\partial r} \left(\frac{1}{r} \frac{\partial r v_r}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 v_r}{\partial \theta^2} + \frac{\partial^2 v_r}{\partial z^2} - \frac{2}{r^2} \frac{\partial v_{\theta}}{\partial \theta} \right] - \frac{\partial p}{\partial r}$

$\rightarrow \frac{\partial p}{\partial r} = 0 \rightarrow p = p_0(z, t)$ (10)

z - momentum

$\rho \left(\frac{\partial v_z}{\partial t} + v_r \frac{\partial v_z}{\partial r} + \frac{v_{\theta}}{r} \frac{\partial v_z}{\partial \theta} + v_z \frac{\partial v_z}{\partial z} \right) = \mu \left[\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial v_z}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 v_z}{\partial \theta^2} + \frac{\partial^2 v_z}{\partial z^2} \right] - \frac{\partial p}{\partial z}$

$\frac{\partial p}{\partial z} = \frac{\partial p_0}{\partial z} = \mu \left[\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial v_z}{\partial r} \right) \right] = f_2(r, t) = P^v$

~~$p_0 = f_2(r)z + p_1(t) = p = p(z, t)$~~
 $p = P^v z + p_1(t)$ where P^v is a global constant

Find v_z

$\frac{\partial}{\partial r} \left(r \frac{\partial v_z}{\partial r} \right) = \frac{P^v}{\mu} r$ $r \frac{\partial v_z}{\partial r} = \frac{P^v}{2\mu} r^2 + A$
 becomes infinite at $r=0$ if $A \neq 0$

$\frac{\partial v_z}{\partial r} = \frac{P^v}{2\mu} r + \frac{A}{r}$
 $v_z = \frac{P^v}{4\mu} r^2 + B$

$v_z = 0$ at $r=R$ $\frac{P^v}{4\mu} R^2 + B = 0 \rightarrow B = -\frac{P^v}{4\mu} R^2$

$v_z = -\frac{P^v}{4\mu} (R^2 - r^2)$
 parabolic

$v_{zmax} = -\frac{P^v}{4\mu} R^2$
 $v_z = v_{zmax} \left(1 - \frac{r^2}{R^2} \right)$

mass flow / unit time

$$\dot{m} = \int \rho \vec{v} \cdot \vec{n} \, dA$$

$$= \iiint \rho v_z \, r \, dr \, d\theta \, dz = 2\pi \int_0^R v_z r \, dr \, dz$$

$$= 2\pi \rho R^2 \int_0^1 v_{z,max} \left(1 - \frac{r^2}{R^2}\right) \frac{r}{R} \, d\frac{r}{R}$$

$$= 2\pi \rho R^2 v_{z,max} \left(\frac{1}{2} - \frac{1}{4}\right) = \rho \frac{1}{2} v_{z,max} \pi R^2$$

$$v_{ave} = \frac{1}{2} v_{max}$$

$$\dot{m} = \rho v_{ave} \pi R^2$$

$$v_{ave} = -\frac{R^2}{8\mu} \frac{dp}{dz}$$

$$v_{ave} = \frac{1}{2} v_{max}$$

$\frac{dp}{dz}$ a global constant

Head (pressure) loss

$$-\Delta p = \frac{dp}{dz} L = \frac{dp}{dz} L \frac{8\mu}{R^2} v_{ave}$$

$$\frac{64\mu}{D^2} L v_{ave} = \frac{64\mu}{\rho v_{ave} D} \frac{L}{D} \frac{1}{2} \rho v_{ave}^2$$

$$h_f = -\frac{\Delta p}{\rho} = \frac{64\mu}{\rho v_{ave} D} \frac{L}{D} \frac{1}{2} \rho v_{ave}^2$$

$\frac{64}{Re_{0,ave}}$ is called the "friction factor" f .

General cylindrical pipe flows

(59)

Assume $v = w = 0$ $u = u(y, z)$

$$P_{2in} = P_{2in}(x) = \frac{dp}{dx} x + P_0 \quad \frac{dp}{dx} \text{ constant}$$

$$u_x + v_y + w_z = 0$$

$$\frac{D\vec{v}}{Dt} = -\frac{\partial \vec{p}}{\partial y} + \mu \nabla^2 \vec{v}$$

$$\frac{Dw}{Dt} = -\frac{\partial p}{\partial z} + \mu \nabla^2 w$$

$$\frac{du}{dt} + u \frac{du}{dx} + v \frac{du}{dy} + w \frac{du}{dz} = -\frac{dp}{dx} + \mu \nabla^2 u = -\frac{dp}{dx} + \mu \left(\frac{\partial^2 u}{\partial y^2} + \frac{\partial^2 u}{\partial z^2} \right)$$

$$\rightarrow \frac{\partial^2 u}{\partial y^2} + \frac{\partial^2 u}{\partial z^2} = + \frac{1}{\mu} \frac{dp}{dx} = \text{constant, negative}$$

1) $u = -A^2 x^2 - B^2 y^2 + C$ produces constant second order derivatives and $u = 0$ on an ellipse

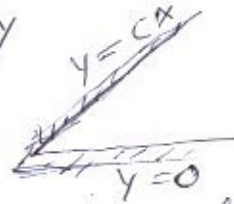


\Rightarrow flow in elliptic pipes of elliptic cross sections

$$\frac{dp}{dx} = -\frac{1}{\mu} (2A^2 + 2B^2)$$

2) $u = -A^2 y^2 + Bxy = -A^2 (y - Cx)y$

\Rightarrow pipe flow near a corner



$$\frac{dp}{dx} = -\frac{1}{\mu} 2A^2$$

most general cubic term with $\nabla^2 = 0$, in suitably related coordinates

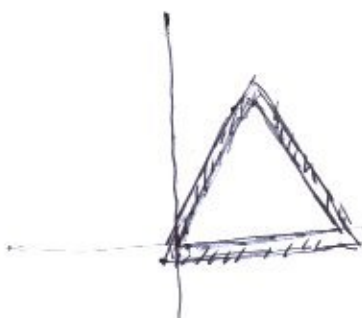
3) $u = Cy(y^2 - 3x^2) - A^2 y(y - \sqrt{3}x) + \dots$

$$= Cy(y - \sqrt{3}x)(y + \sqrt{3}x) - A^2 y(y - \sqrt{3}x) + \dots$$

$$= y(y - \sqrt{3}x) [-A^2 + C(y + \sqrt{3}x)]$$

$$\frac{dp}{dx} = -\frac{1}{\mu} 2A^2$$

Flow in ~~an~~ a pipe with equilateral triangular cross-section



$$\frac{dp}{dx} = -\frac{1}{\mu} 2A^2$$